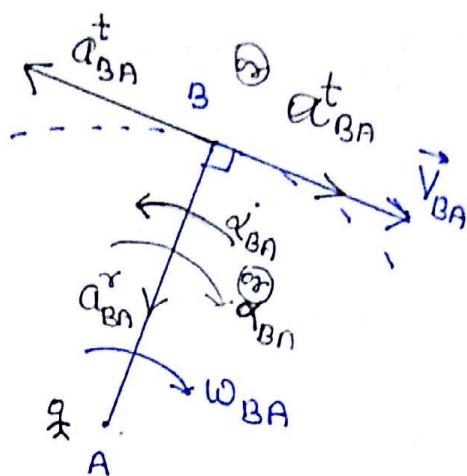


## Acceleration Analysis



Must be present

$$a_{BA}^r = \frac{V_{BA}^2}{BA} = (BA)(\omega_{BA}^2)$$

bcoz of the  
change in  
Velocity dir^n

\* In Circular motion this  
accn  $\neq 0$

\* this is known for every  
dir^n towards centre.

dir^n ( $B \rightarrow A$ )

$$\vec{a}_{BA} = \frac{d}{dt} \vec{v}_{BA}$$

$$a_{BA}^t = (BA) \alpha_{BA}$$

bcoz of the  
change in  
Velocity magnitude

\* only known for  
input line.

\* May or May not be zero.

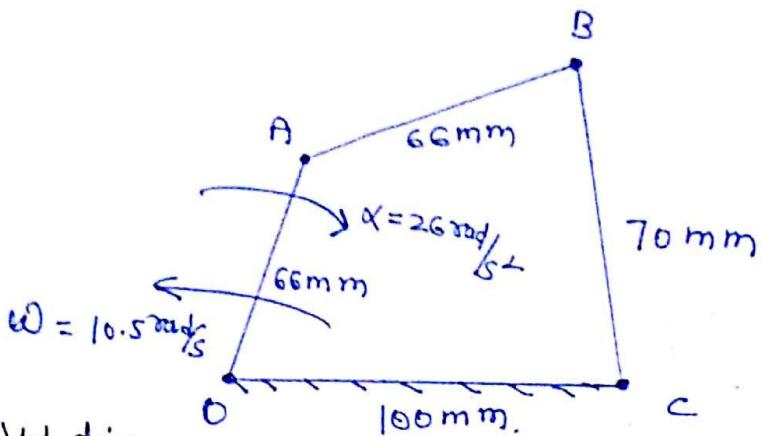
\* Dir^n  $\perp$  to Radial.

$$a^r = r\omega^2$$

$$a^t = r\alpha$$

$$\alpha = \frac{d\omega}{dt}$$

Prob

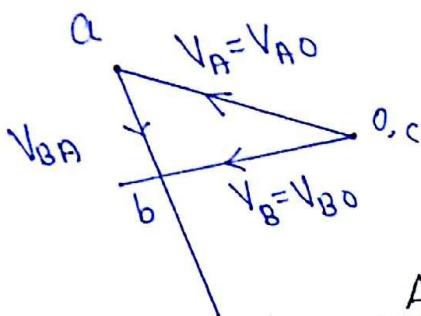


$$\alpha_B = ?$$

$$\alpha_{BA} = ??$$

$$\alpha_{BC} = 2?$$

Scale Vel dia

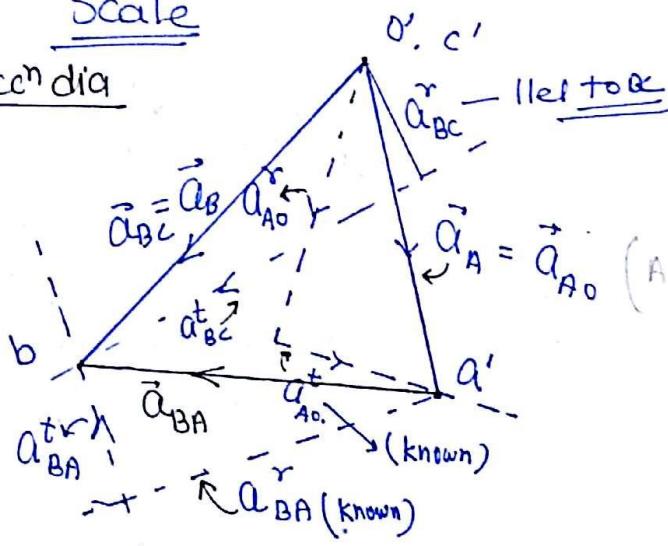


$$V_{AO} = (0.066) \times 10.5$$

$$V_{AO} =$$

m/s → Scale and draw V-dig

Scale Acc dia



After Velocity dig

$$a_{AO}^r = \frac{V_{AO}^2}{AO} \quad \checkmark$$

$$a_{BA}^r = \frac{V_{BA}^2}{BA} \quad \checkmark$$

$$a_{BC}^r = \frac{V_{BC}^2}{BC} \quad \checkmark$$

$$a_{AO}^t = (AO) \times \alpha_{AO} \quad (\text{Given})$$

$\perp \text{ to } a^r$

measure

d-scale & find

$$\alpha_{BC} \& \alpha_{BA}$$

Point w.r.t.

A O

Procedure

$a_{AO}^r \rightarrow \text{known}$   
 $a_{AO}^t \rightarrow \text{known}$ .

B

A

$a_{BA}^r = \text{known}$   
 $a_{BA}^t = a_{BA}^r \text{ (unknown)}$   
 $\perp \text{ to radial}$

B

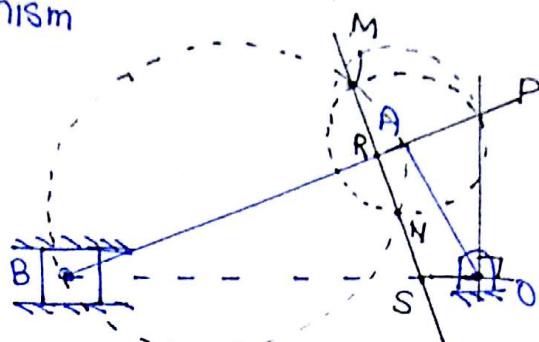
C

$a_{BC}^r = \text{known}$   
 $a_{BC}^t = a_{BC}^r \text{ (unknown)}$   
 $\perp \text{ to radial}$

## Klein's Construction:

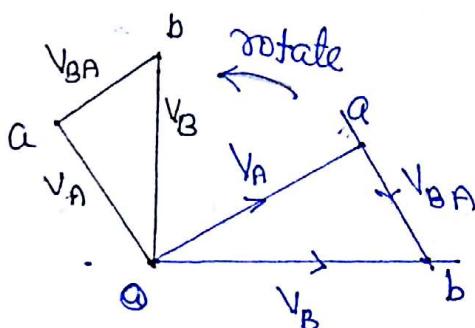
This construction is only applied in Basic Single Slider Crank Mechanism when  $\alpha_{\text{crank}} = 0$

Given  $\omega_{\text{crank}}$



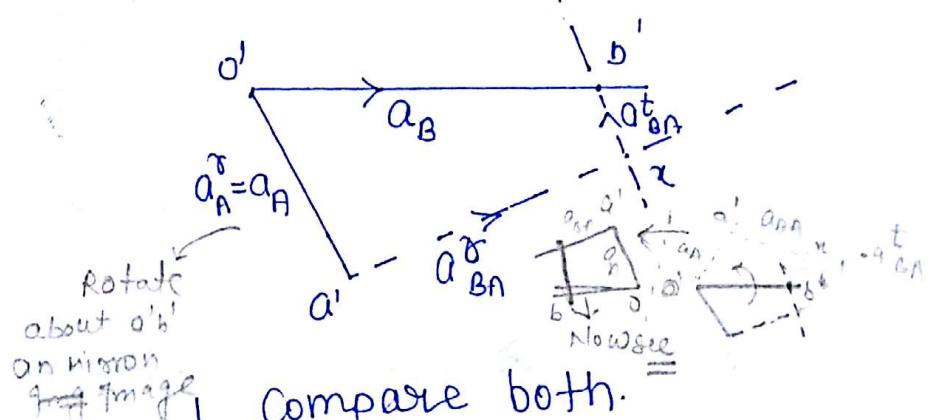
$\triangle OAP$  (Velocity  $\Delta$ )

Velocity diagram (Rough)



$\triangle OARS$  (Acc<sup>n</sup>  $\Delta$ )

Acc<sup>n</sup> diagram (Rough)



Compare both

$$\frac{v_A}{OA} = \frac{v_B}{OP} = \frac{v_{BA}}{AP} = \omega_{\text{crank.}}$$

↓  
Given.

Compare both.

$$\frac{a_A}{OA} = \frac{a_{BA}^t}{RS} = \frac{a_B}{OS} = \frac{a_{BA}^r}{AR} = \omega_{\text{crank.}}^2$$

### Note

① If point O and P coincide  $\Rightarrow OP = 0$

$$V_B = 0$$

Slider at extreme position

② If point O and S coincide  $\Rightarrow OS = 0$

$$\alpha_B = 0$$

$$V_B \rightarrow \text{maximum}$$

Slider at mid position

③ If point A and R coincide  $\Rightarrow AR = 0$

$$a_{BA}^{\infty} = 0$$

$\Rightarrow$  (C.R.) - pure translation.

4. If point R & S coincide  $\Rightarrow RS = 0$

$$a_{BA}^t = 0$$

$$\Rightarrow (\dot{\theta}_{BA})(\ddot{\theta}_{BA}) = 0$$

$$\dot{\theta}_{BA} = 0$$

$$\omega_{BA} = \text{Constant}$$

Connecting Rod having constant angular velocity.

## Coriolis Accen: ( $a^c$ )

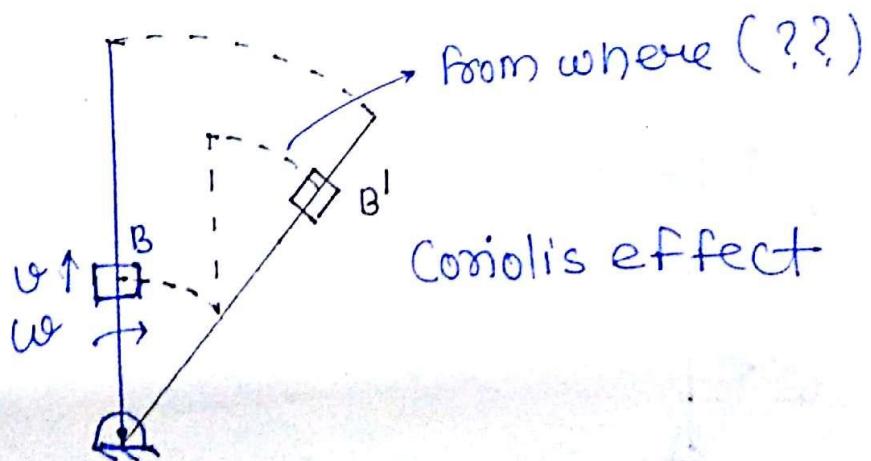
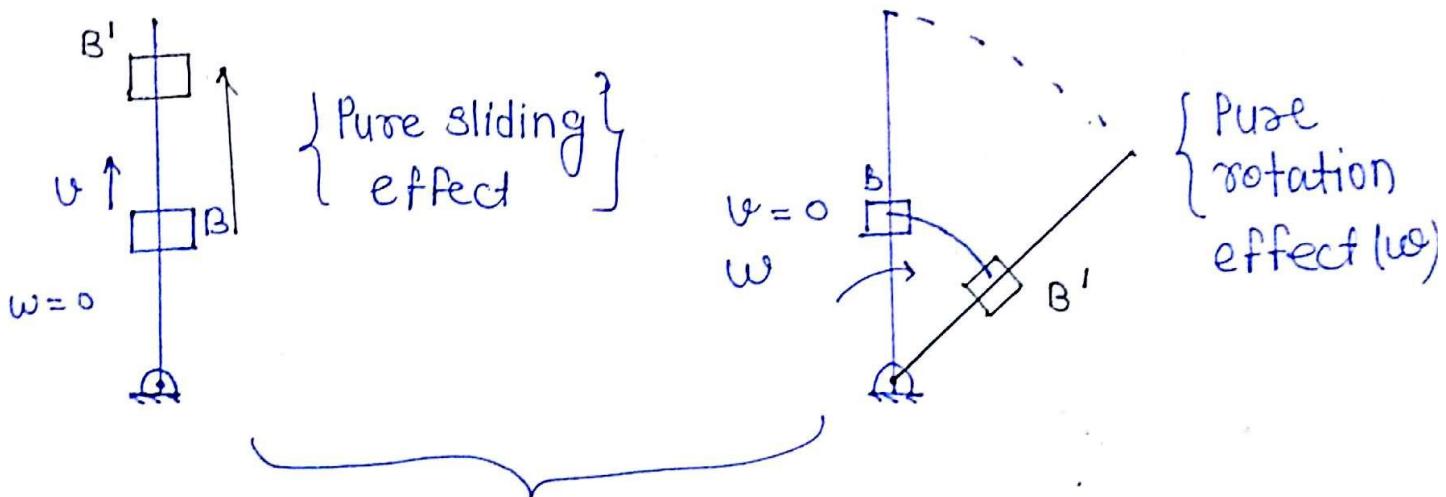
This accen will always be associated with the slider, when the slider is sliding on the Rotating Body.

The magnitude will be:-

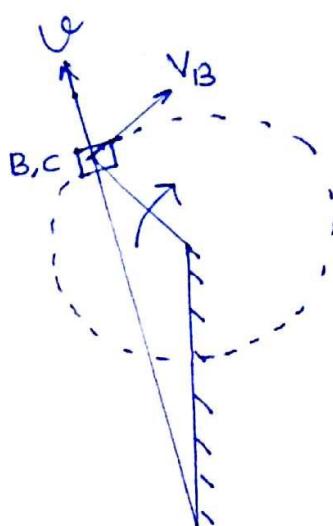
$$a^c = 2v\omega$$

where  $v$  - sliding Velocity of Slider

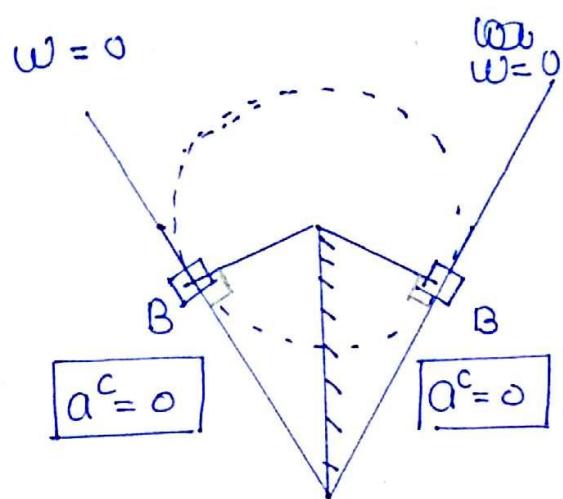
$\omega$  - Angular Velocity of Body on which slider is sliding.



In crank slotted mechanism.

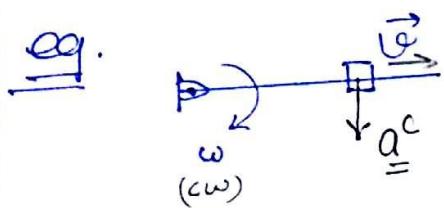


$a^c$  will  
exists

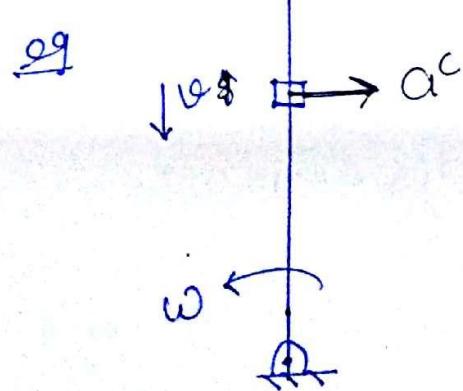
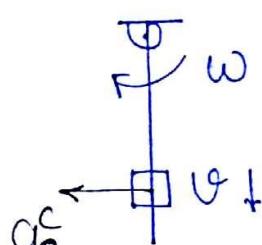


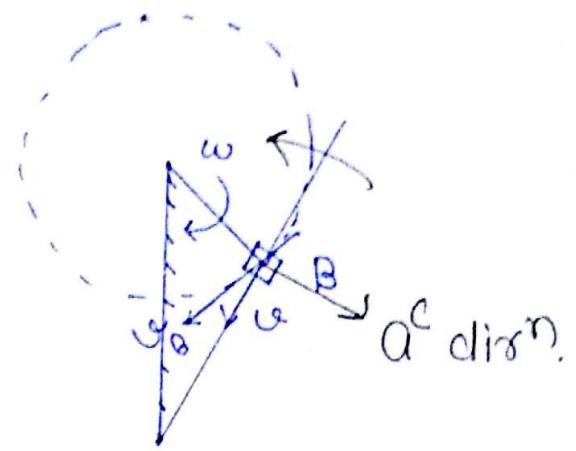
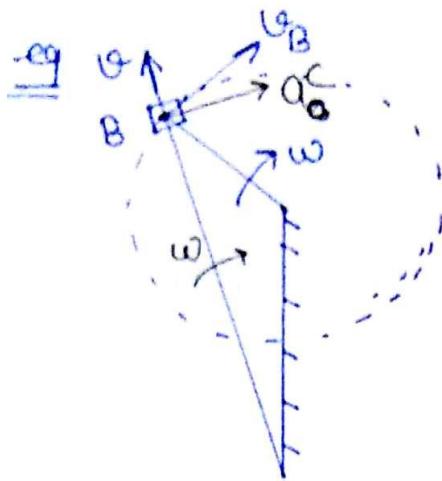
A + 4 position from  $\infty$  position  
 $a^c$  will be zero.

- Direction of  $a^c$
- (i) Take the sense of  $\omega$
  - (ii) Rotate the  $\vec{v}$  in that sense by  $90^\circ$



eg

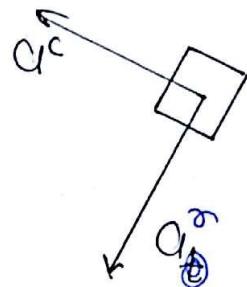
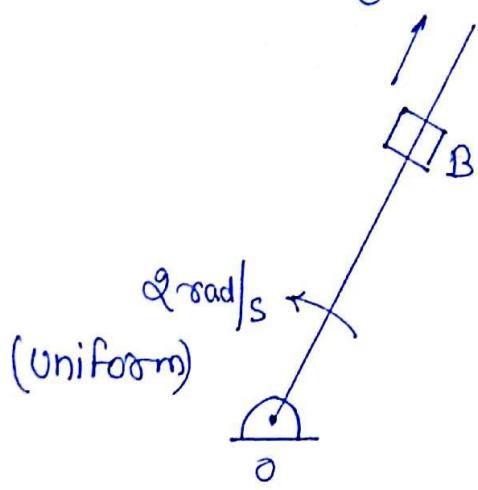




Problem:-

$$\omega = 0.75 \text{ m/s}$$

Find the accn of Slider



$$a^c = \omega v \omega = \omega (0.75) \times 2 = 3 \text{ m/s}^2$$

$$a_B^c = \omega^2 r = (1)^2 (2) = 4 \text{ m/s}^2$$

$$a_{\text{net}} = \sqrt{3^2 + 4^2} = 5 \text{ m/s}^2$$