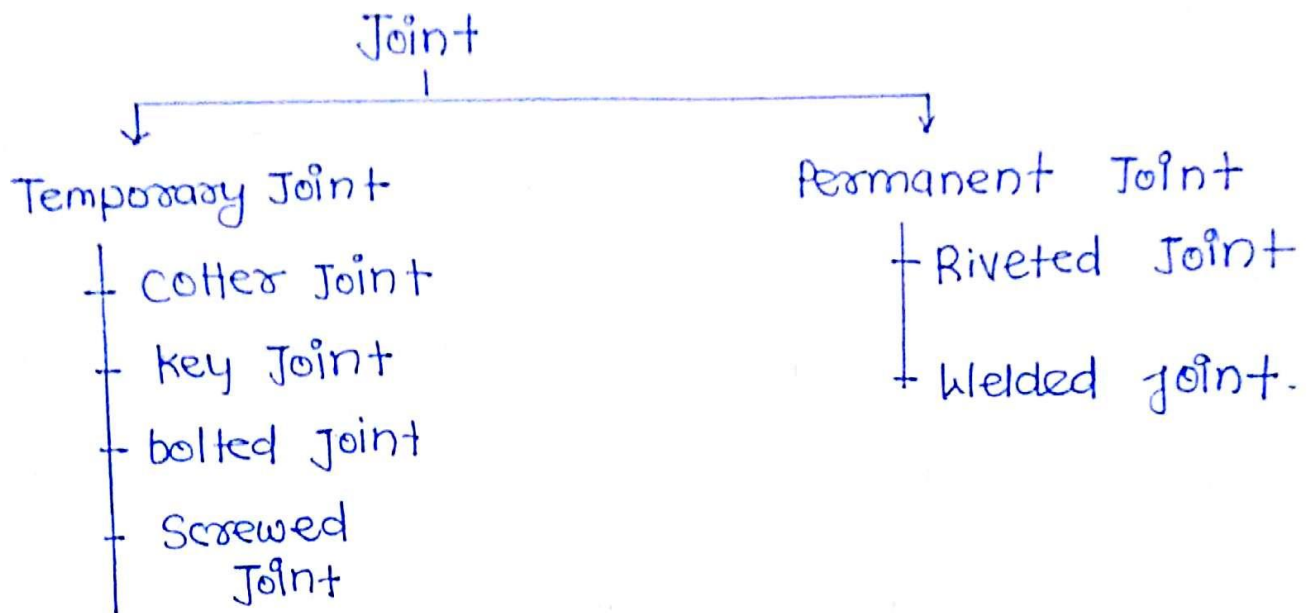
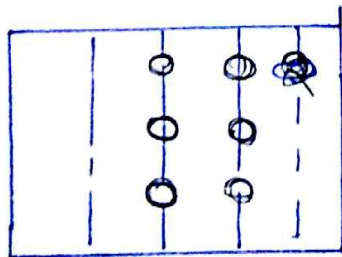
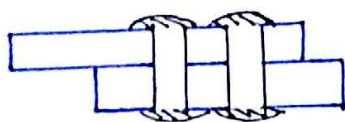
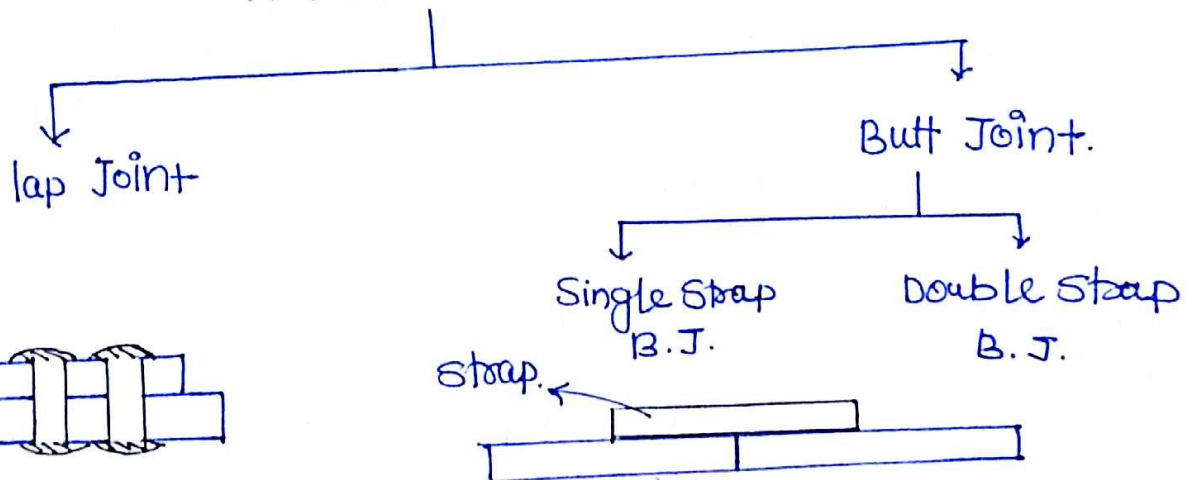


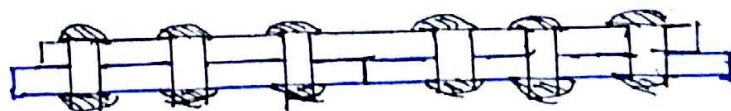
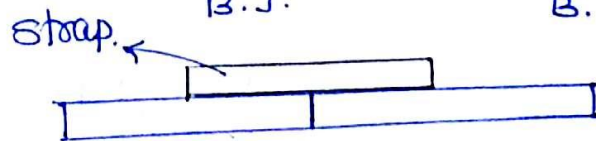
* Riveted Joint *



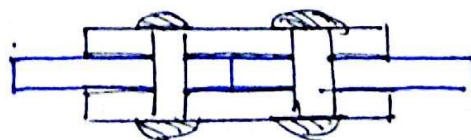
Riveted Joint



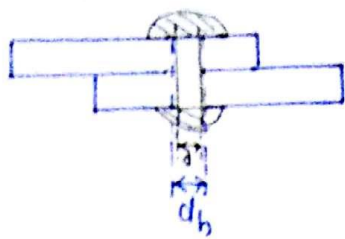
Double row lap joint
D.R.L.J.



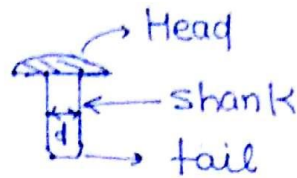
Double row single strap B.J.



Single row double strap. B.J.



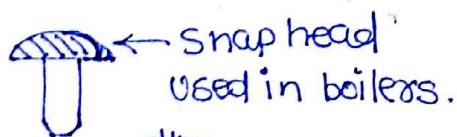
$d_h = \text{hole dia}$



Riveted = shank dia 'd'

- For the safe design of rivet shank dia 'd' will be taken in consideration and for the safe design of plate hole dia 'd_h' will be taken in consideration.

Riveted joint = Rivet + Plate



← Snap head
Used in boilers.
~~Strong~~
Strong in fatigue
& Impact



Pan head
Strength max^m
Manufacturing difficult
Ship builds, boilers.
Coesive ~~str~~ resistance is more.



← Counter sunk
Manf. simple
(under water work)



Flat head
Thin plate or
structural work.

By Unwin's Formula

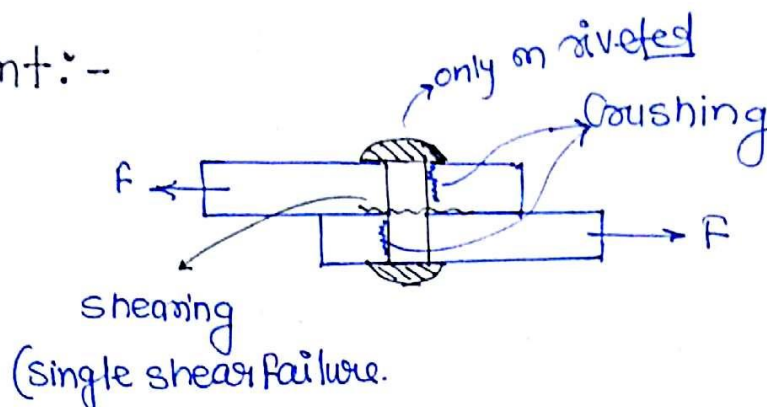
shank ~~joint~~ diameter $d = 6 \sqrt{t}$

$\downarrow$ mm $\uparrow$ mm

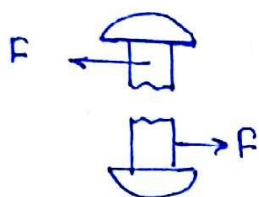
t = thickness of plate

Types of failure:-

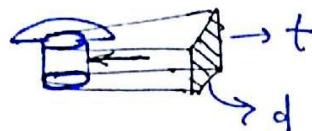
Lap Joint:-



Lap joint
 $\Rightarrow$ shear &
 $\Rightarrow$ crushing



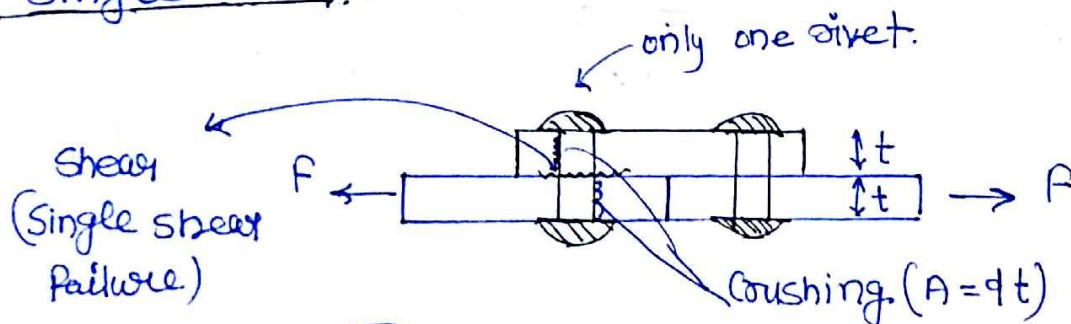
$$(\tau_{ind})_{\text{shear}} = \frac{F}{\frac{\pi}{4} d^2}$$



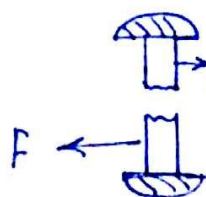
$$(\sigma_{ind})_{\text{crushing}} = \frac{F}{d t} \leftarrow \text{projected area.}$$

Butt Joint:-

Single strap:-



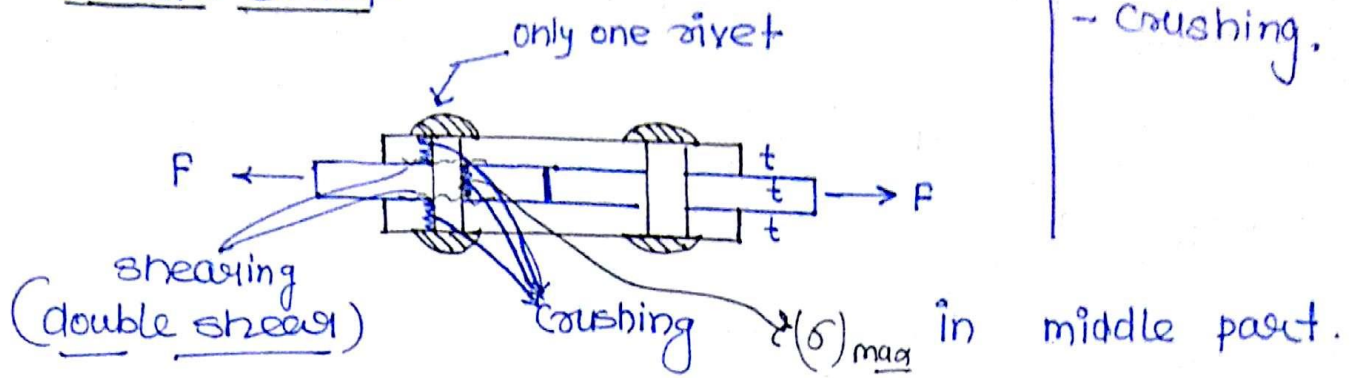
single strap
 $\Rightarrow$ shear &
 $\Rightarrow$ crushing



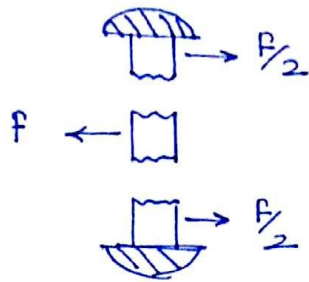
$$\tau_{ind} = \frac{F}{\frac{\pi}{4} d^2}$$

$$(\sigma_{ind})_{\text{crushing}} = \frac{F}{d t}$$

Double strap



double strap
→ double shear
→ crushing.



$$\tau_{ind} = \frac{F}{2 \cdot \frac{\pi}{4} d^2}$$

crushing

$$(\sigma_{ind})_{crush\ max} = \frac{F}{dt}$$

$(\tau_{ind})_{shear} = \frac{F}{k \cdot \frac{\pi}{4} d^2}$
$(\sigma_{ind})_{crush} = \frac{F}{dt}$

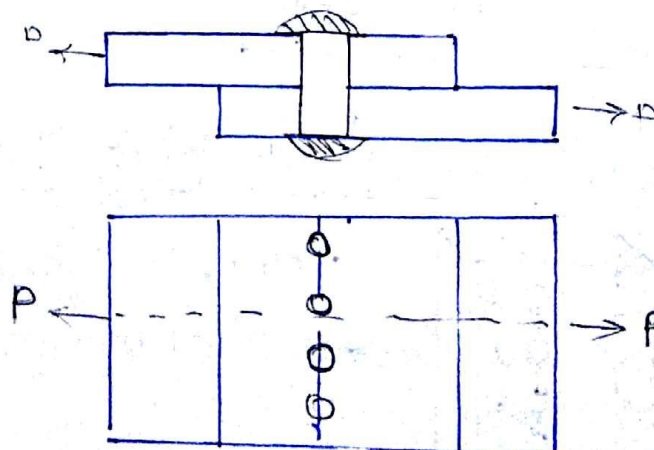
k = 1 single shear

k = 2 double shear

for all riveted joint.

Design of Riveted Joint: ⇒

Case-1:- Finite Riveting :-



$$F_{rivet} = \frac{P}{4}$$

load

Shear design of Rivet:-

$$\tau_{ind} = \frac{P}{4k \frac{\pi}{4} d^2}$$

Safe condⁿ

$$\frac{P}{4 \cdot k \cdot \frac{\pi}{4} d^2} \leq \sigma_{per.}$$

Shear strength
of Riveted
joint

$$P_{max} = 4 \cdot k \cdot \frac{\pi}{4} d^2 \sigma_{per.}$$

↓
no. of rivets

Crushing design of Rivet.

$$\sigma_{ind} = \frac{P}{4 \cdot d \cdot t}$$

Safe condⁿ

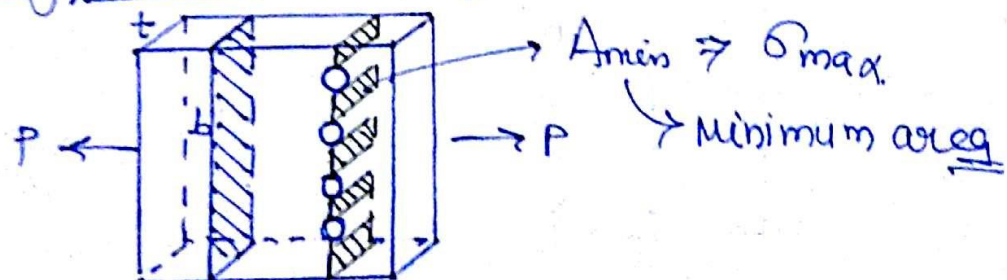
$$(\sigma_{ind})_{crush} \leq \sigma_{per.}$$

$$\frac{P}{4dt} \leq \sigma_{per.}$$

Crushing
strength of Rivet.

$$P_{max} = 4dt \sigma_{per.}$$

★ Tearing/tension strength of joint/Plate



$$(\sigma_{ind})_{\max}^{\text{tearing}} = \frac{P}{(b - 4d_h)t}$$

Consider hole
dia. d_h

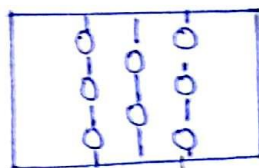
safe condⁿ.

$$(\sigma_{ind})_{\max} \leq \sigma_{per.}$$

$$\frac{P}{(b - 4d_h)t} \leq \sigma_{per.}$$

tearing
Strength of
Rivet.

$$P_{\max} = (b - 4d_h)t \cdot \sigma_{per.}$$



→ take min area.

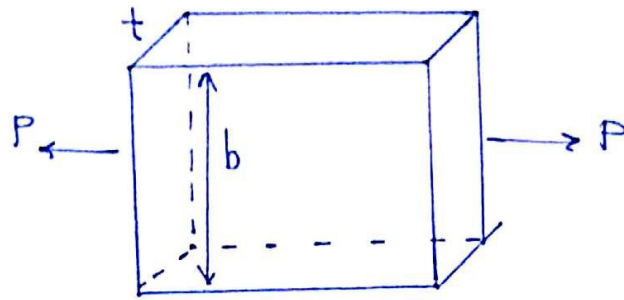
Actual strength of Riveted joint =

$$= \min \text{ of } \left[(P_{\max})_{\text{shearing}}, (P_{\max})_{\text{crush}}, (P_{\max})_{\text{tearing}} \right]$$

Actual strength of Rivet =

$$= \min \text{ of } \left[(P_{\max})_{\text{shear}}, (P_{\max})_{\text{crush}} \right]$$

Strength of solid plate :-



$$\sigma_{ind} = \frac{P}{bt}$$

Best possible joint.

$$P_{max} = bt \sigma_{per}$$

Solid

$$\eta_{shearing} = \frac{(P_{max})_{shear}}{(P_{max})_{solid}} = \frac{4 \cdot k \cdot \frac{\pi}{4} d^2 \cdot \tau_{per} \xrightarrow{\text{Rivet}}}{bt \sigma_{per} \xrightarrow{\text{plate}}}$$

$$\eta_{crushing} = \frac{(P_{max})_{crush}}{(P_{max})_{solid}} = \frac{4 d t [\sigma_{per}] \xrightarrow{\text{Rivet}}}{bt [\sigma_{per}] \xrightarrow{\text{plate}}}$$

$$\eta_{tearing} = \frac{(P_{max})_{tearing}}{(P_{max})_{solid}} = \frac{(b - 4d_h)t \cdot \sigma_{per} \xrightarrow{\text{plate}}}{bt \sigma_{per} \xrightarrow{\text{plate}}}$$

$$\eta_{tearing} = 1 - \frac{4d_h}{b}$$

$$\eta_{actual} = \min \text{ of } [\eta_{shearing}, \eta_{crushing}, \eta_{tearing}]$$

Q.3.12

Q2 =

$$\underline{3.12} \quad (P_{\max})_{\text{crush}} = 3dt \sigma_{\text{per}} = 3 \times 10 \times 5 \times$$

$$3.13 \quad (P_{\max})_{\text{tearing}} = (w - 3dh) \times t \times \sigma_{\text{per}}$$

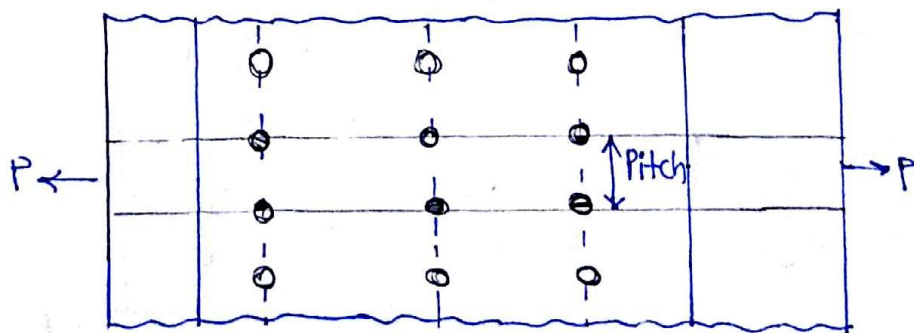
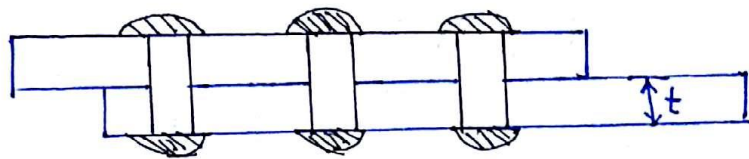
Q.3.14

$$P = 19 \text{ kN}$$

$$\frac{200 \times 10^6}{2} \geq \frac{19 \times 10^3}{3 \times \frac{\pi}{4} \times d^2}$$

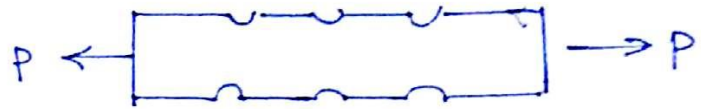
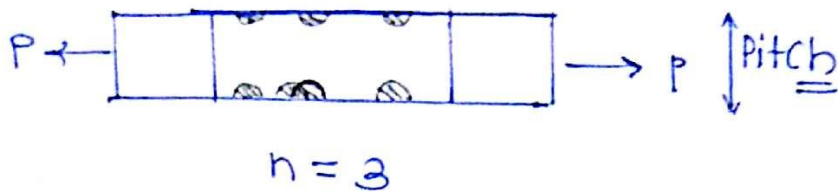
$$d = 10 \text{ mm}$$

Case-2 Infinite Riveting



$n = \text{no. of rivet per pitch length}$

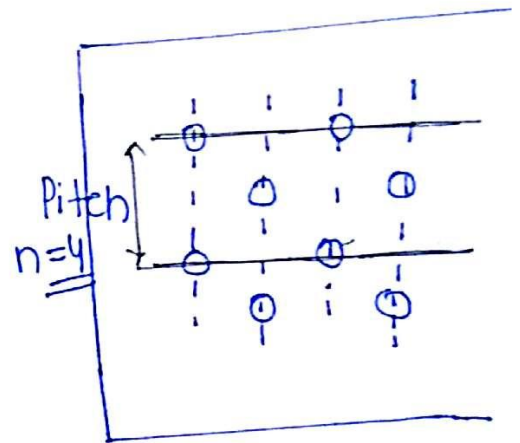
$$n = 3$$



$$(P_{\max})_{\text{shear}} = n \cdot k \cdot \frac{\pi d^2}{4} \tau_{\text{per.}}$$

$$(P_{\max})_{\text{crush}} = n d t \sigma_{\text{per.}}$$

$$(P_{\max})_{\text{tearing}} = (p - d_h) t \cdot \sigma_{\text{per.}}$$



$$(P_{\max})_{\text{solid}} = p \cdot t \cdot \sigma_{\text{per.}}$$

$$\eta_{\text{shear}} = \frac{n \cdot k \cdot \frac{\pi d^2}{4} \cdot \tau_{\text{per.}}}{p \cdot t \cdot \sigma_{\text{per.}}} \quad ; \quad \eta_{\text{crush}} = \frac{n d t \sigma_{\text{per.}}}{p t \sigma_{\text{per.}}}$$

$$\eta_{\text{tearing}} = \frac{(p - d_h) t \sigma_{\text{per.}}}{p t \sigma_{\text{per.}}} \quad \underline{\underline{\eta_{\text{imp}}}}$$

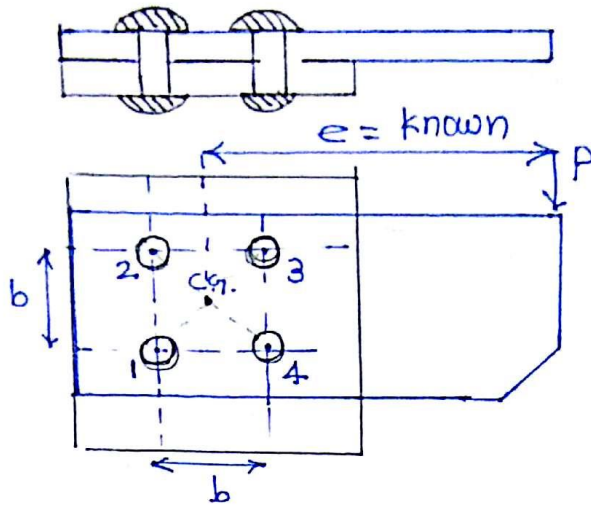
Case 3

$$\underline{\underline{\eta_{\text{imp}}}} \quad \eta_{\text{tearing}} = 1 - \frac{d_h}{p}$$

p - pitch

$\downarrow$

Case-3



Step ① Find out the C.G. of Group of rivets.

$$\bar{x} = \frac{x_1 + x_2 + x_3 + x_4}{n}, \quad \bar{y} = \frac{y_1 + y_2 + y_3 + y_4}{n}$$

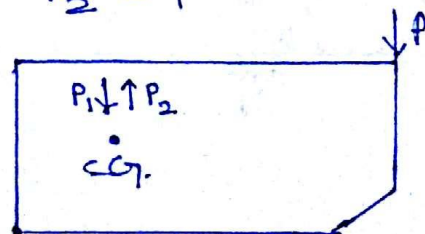
$$\bar{x} = \frac{0 + 0 + b + b}{4} = \frac{b}{2}, \quad \bar{y} = \frac{0 + b + b + 0}{4} = \frac{b}{2}$$

Step ② Find out eccentricity

↪ dist. b/w line of action to C.G. of group of rivet.

Step ③ Apply two equivalent & opp. forces

P_1 & P_2 Passing through C.G. of group of rivet, parallel to the load such that $P_1 = P_2 = P$



Step ④

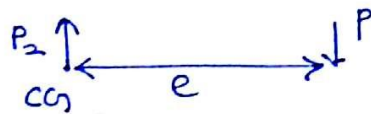
effect of P_1



$P_1/4$ = primary shear force.

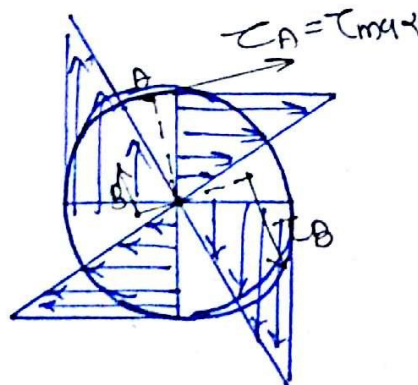
As P_1 passes through C.G. of the group to the rivet, it results primary shear force induced of equal magnitude in each rivet.

effect of P_2 & P

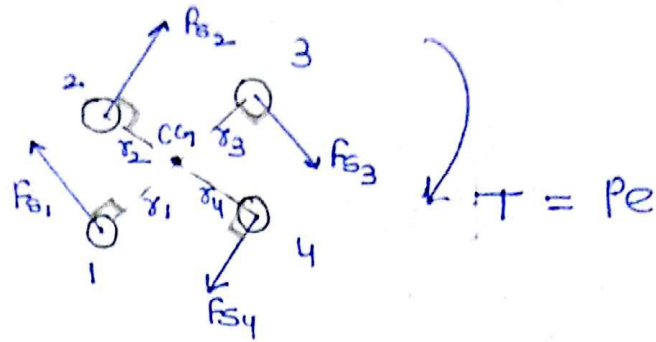


P_2 & P causes a constant twisting couple of ' Pe ' in CW dirⁿ about the CG which results twisting in rivets.

twisting in shaft $\tau_{\text{torque}} \propto r$



(C) $\text{torque} \propto \gamma \Rightarrow F_s \propto \gamma \Rightarrow \text{Secondary shear force}$



$$F_{s1} \cdot \gamma_1 + F_{s2} \cdot \gamma_2 + F_{s3} \cdot \gamma_3 + F_{s4} \cdot \gamma_4 = P.e$$

$$F_s \propto \gamma$$

$$\frac{F_{s1}}{\gamma_1} = \frac{F_{s2}}{\gamma_2} = \frac{F_{s3}}{\gamma_3} = \frac{F_{s4}}{\gamma_4} \dots$$

$$\frac{F_{s1}}{\gamma_1} \left[\gamma_1^2 + \frac{F_{s2}}{F_{s1}} \gamma_1 \cdot \gamma_2 + \frac{F_{s3}}{F_{s1}} \gamma_1 \cdot \gamma_3 \dots \right] = P.e$$

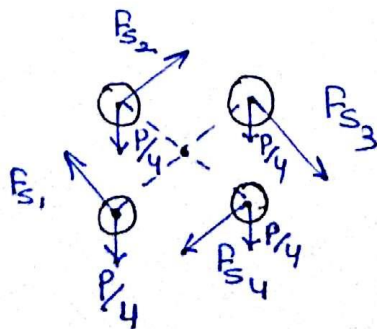
$$\frac{F_{s1}}{\gamma_1} \left[\gamma_1^2 + \gamma_2^2 + \gamma_3^2 + \gamma_4^2 \right] = P.e$$

$F_{s1}, F_{s2}, F_{s3}, F_{s4}$ known.

if $\gamma_1 = \gamma_2 = \gamma_3 = \gamma_4$

$$F_{s1} = F_{s2} = F_{s3} = F_{s4} = \frac{P.e}{4\gamma}$$

Combined effect (P_1 & Couple $P_2 P$)

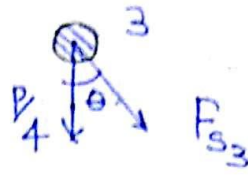


critical rivet
or worst loaded rivet

$$R^2 = p^2 + q^2 + 2pq \cos \theta$$

$$\theta \downarrow \rightarrow R \uparrow$$

Critical rivets are 3 & 4.



$$R_3 = \sqrt{\left(\frac{P}{4}\right)^2 + (F_{S_3})^2 + 2 \times \frac{P}{4} \times F_{S_3} \cos \theta}$$

→ this is the maximum shear.

$$\tau_{\max} = \frac{R_3}{\frac{\pi}{4} d^2}$$

Safe Condⁿ

$$\tau_{\max} \leq \tau_{\text{per.}}$$

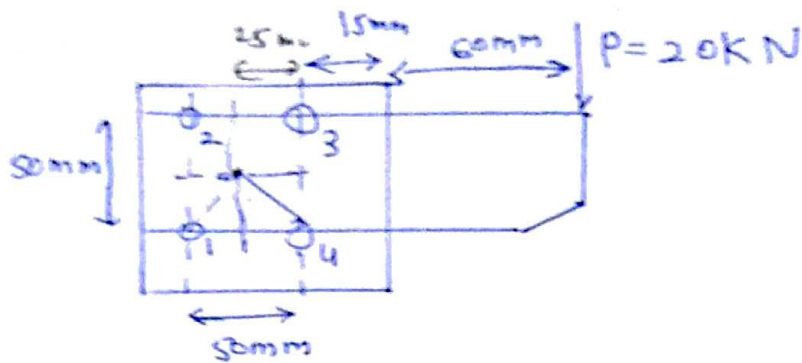
$$\frac{R_3}{\frac{\pi}{4} d^2} \leq \tau_{\text{per.}}$$

$$d \geq ?$$

~~τ_3~~
max

known.

Question



$$\tau_{\text{per}} = 70 \text{ MPa}$$

$$C.G. = (25, 25)$$

$$P_1 = P_2 = P_3 = P_4 = \frac{20}{4} = 5 \text{ kN}$$

$$\frac{F_1}{r_1} (r_1^2 + r_2^2 + r_3^2 + r_4^2) = 20 \times 10^3 \times \frac{100}{100}$$

$$\frac{F_1}{25\sqrt{2}} ((25\sqrt{2})^2 + 4 \times 4) \times 10^{-3} = 20 \times 100$$

$$F_1 = 14.14 \text{ kN}$$

$$\frac{F_1}{r_1} = \frac{F_3}{r_3} \Rightarrow F_3 = F_1$$

$$R_3 = \sqrt{(5000)^2 + (14142)^2} + (14142) \times \frac{1}{\sqrt{2}}$$

$$\frac{R_3}{\pi d^2 / 4} \leq \tau_{\text{per}}$$

$$R_3 = 18.02 \text{ kN}$$

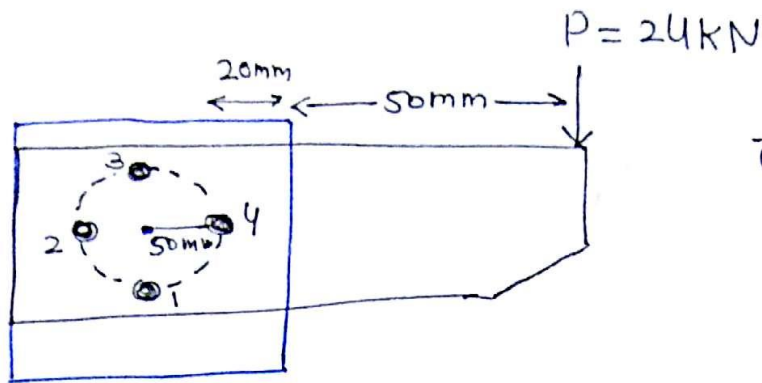
$$d \geq \sqrt{\frac{4 R_3}{\pi \tau_{\text{per}}}}$$

$$\Rightarrow \sqrt{\frac{70 \times 10^6 \times \pi}{4 \times 14142 \times 18.02 \text{ kN}}}$$

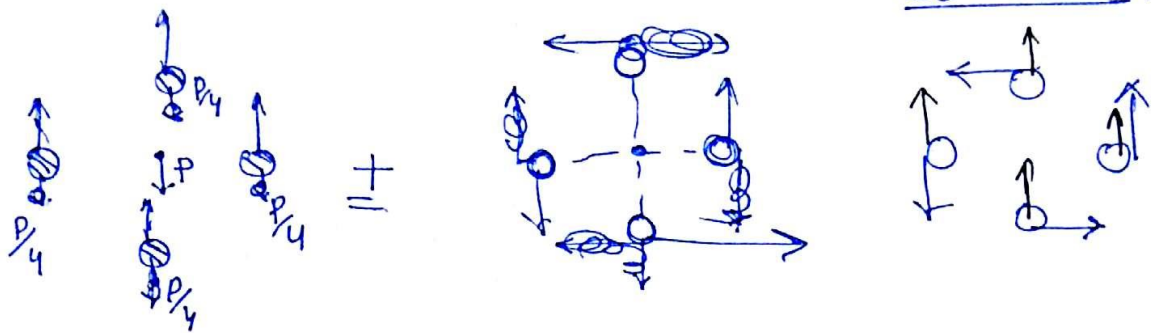
$$d \geq 18.1 \text{ mm}$$

$$d = 19 \text{ mm}$$

Ques



$$\tau_{per} = 75 \text{ MPa}$$



$$\frac{F}{r_i} (r_1^2 + r_2^2 + r_3^2 + r_4^2) = 24 \times 120$$

$$4F(50) = 24 \times 120$$

$$F = 14.4 \text{ kN}$$

$$R_{net} = \sqrt{\left(\frac{24}{4}\right)^2 + (14.4)^2} = 27.98$$

$$(R_{net})_y = 14.4 + 6 = 20.4 \text{ kN}$$

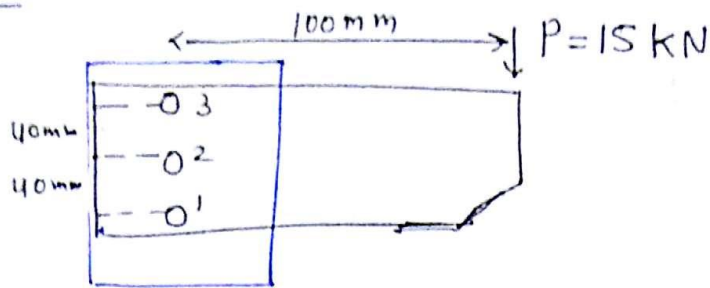
$$\frac{(R)_{max}}{\frac{\pi}{4} d^2} \leq \tau_{per}$$

$$d \geq \sqrt{\frac{R \times 4}{\pi \times \tau}} = \sqrt{\frac{20.4 \times 4 \times 10^3}{\pi \times 75}}$$

$$d \geq 18.6$$

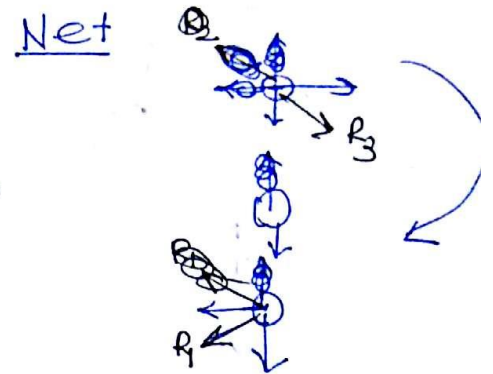
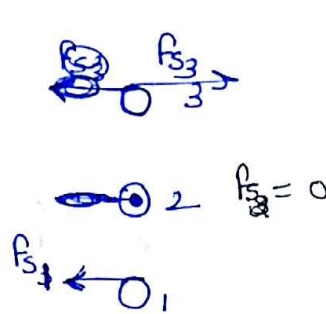
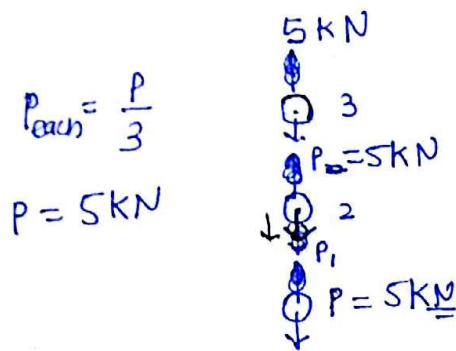
$$d = \underline{\underline{19 \text{ mm}}}$$

Question



$$\tau_{per} = 75 \text{ MPa}$$

- (i) Find out the critical rivet
- (ii) Resultant force on each rivet
- (iii) Safe dia for rivet.



$$(F_{S3} \times 40) \times 2 = 15 \times 100$$

$$F_{S3} = 18.75 \text{ kN}$$

$$(R)_{net} = R_1 = R_3 = 19.40 \text{ kN} \quad R_2 = 5 \text{ kN}$$

$$d = \sqrt{\frac{19.40 \times 410}{\pi \times 75}}$$

$$d \geq 18.15$$

$$d = \underline{\underline{19 \text{ mm}}}$$

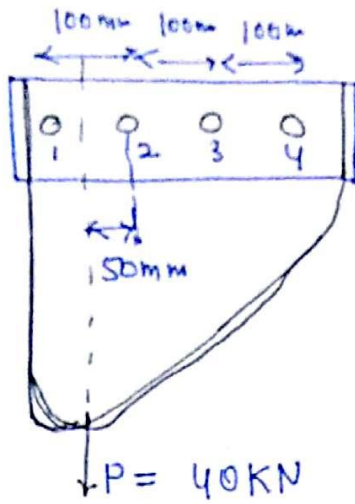
$$(i) 1, 3$$

$$(ii) R_1 = R_3 = 19.40 \text{ kN}$$

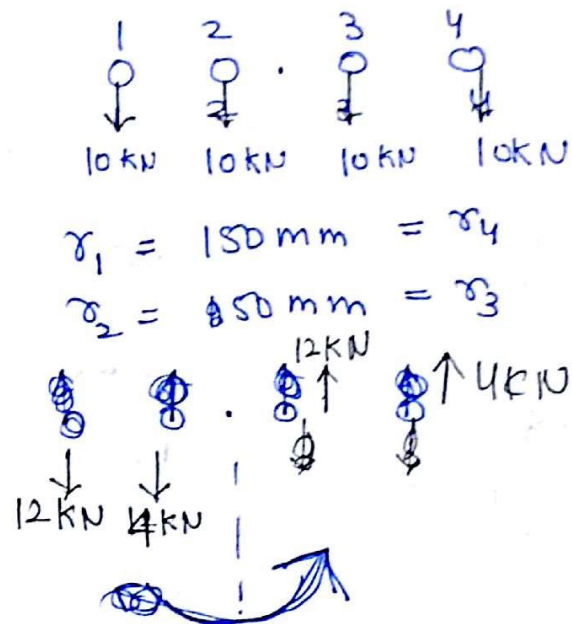
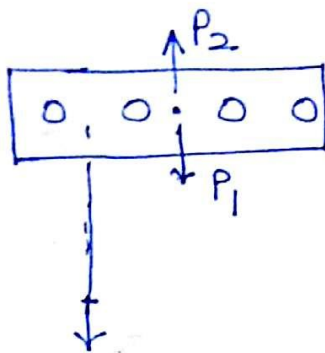
$$(iii) d = 18.15 \text{ mm}$$

$$= \underline{\underline{19 \text{ mm}}}$$

Que



$$\tau_{\text{per}} = 80 \text{ MPa}$$



$$T = P.e = 40 \times 150 \text{ kN mm}$$

$$\frac{F_{s1}}{r_1} (r_1^2 + r_2^2 + r_3^2 + r_4^2) = P.e.$$

$$\frac{F_{s1}}{150} (2 \times 150^2 + 2 \times 50^2) = 40 \times 150$$

$$F_{s1} = 12 \text{ kN} \quad F_{s2} = 12 \times \frac{1}{3} = 4 \text{ kN}$$

$$F_{s3} = 4 \text{ kN} \quad F_{s4} = 12 \text{ kN}$$

Net

$$R_1 = 20 \text{ kN}$$

$$R_2 = 14 \text{ kN}$$

$$R_3 = 10 \text{ kN} - 2 \text{ kN}$$

$$R_4 = 20 \text{ kN} - 6 \text{ kN}$$

Critical

R4

$$d \geq \sqrt{\frac{22 \times 4 \times 10^3}{\pi \times 80}}$$

$$d = 18.74$$

$$d = 19 \text{ mm}$$