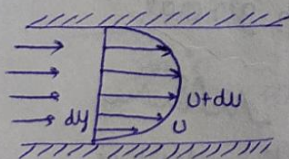


Lecture 13
8/04/19

Flow through Pipes ⊕ Chapter 10

Pipe Flow :- (imp. topic for gate) It is closed conduit generally having circular cross-section. **Note** the cross-section may be of different other shape such as $\square$ $\triangle$ $\square$ e.t.c.
Note: For other shape cross-section of closed conduit, terminology like dust is used.

Pipe Flow:- When the flow completely fills the cross-section of pipe and the flows are considered as pipe flow.



$\tau = \text{exist}$

$$F_{vis} = \tau \cdot A$$

$$Re = \frac{F_{inertia}}{F_{viscous}}$$

$$Re = \frac{\rho \cdot V \cdot L}{\mu}$$

$L = \text{characteristic dimension}$

Characteristic $\Rightarrow$ hydraulic Diameter (D_{TH})
Dimension

$$Re = \frac{\rho \cdot V \cdot D_{TH}}{\mu}$$

$$D_{TH} = \frac{4A}{P}$$

$P = \text{Wetted Perimeter}$

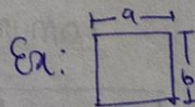
For Circular,

$$D_{TH} = \frac{4A}{P}$$

$$= \frac{4 \times \pi \times D^2}{4} \div \pi D$$

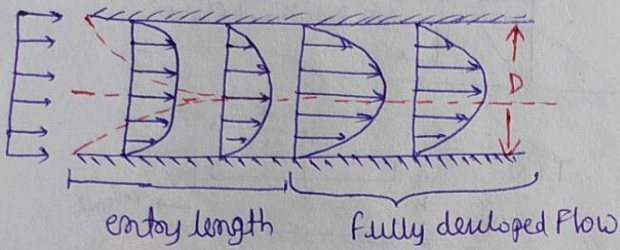
$$D_{TH} = D$$

$$Re = \frac{\rho \cdot V \cdot D}{\mu}$$

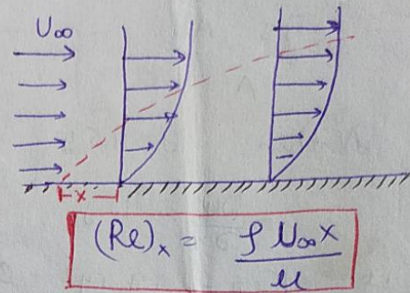


FLOW physics (characteristic Dimension)

Pipe Flow



Flow over Flat Plate



Head losses:-

- Major head losses
- Minor head losses

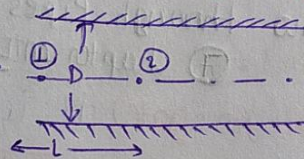
Major Head losses

① Darcy weisbach eqn :-

(It is used for only closed channel)

$$V = \frac{Q}{\frac{\pi}{4} D^2}$$

$$h_f = \frac{fL}{2gD} \left[\frac{Q}{\frac{\pi}{4} D^2} \right]^2 = \frac{8Q^2 \cdot f \cdot L}{\pi^2 g D^5}$$



$V \rightarrow$ mean velocity of pipe flow

$$h_f = \frac{fL \cdot V^2}{2gD}$$

① f = friction factor

$$\textcircled{2} f = 4f'$$

③ f' = [coefficient of friction]

② For pipe flow (circular c/s)

$Re < 2000$ [laminar]

$Re > 4000$ [turbulent]

laminar

$$f' = \frac{16}{Re}$$

and

$$f = \frac{64}{Re}$$

turbulent

$$f' = \frac{0.079}{Re^{\frac{1}{4}}}$$

and

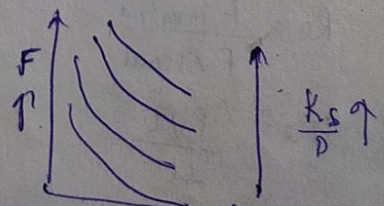
$$f = \frac{0.316}{Re^{\frac{1}{4}}}$$

In general,

$h_f \propto \frac{K_s}{D}$

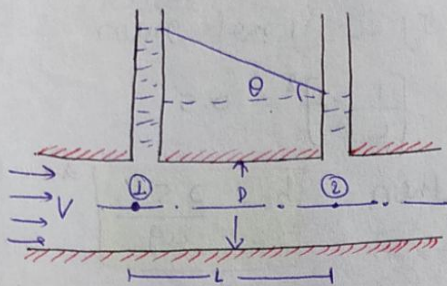
K_s = surface roughness

Mody's chart $\rightarrow$



- Chezy's Formula :-

(It is used for open as well as close channel)



V = mean velocity of pipe flow

$$V = C \sqrt{mi}$$

$$V = C \sqrt{\frac{D}{4} \cdot \frac{h_f}{L}}$$

$$V^2 = C^2 \frac{D}{4} \cdot \frac{h_f}{L}$$

$$\left[h_f = \frac{4(L \cdot V^2)}{C^2 D} \right]$$

m = hydraulic mean depth

$$m = \frac{A}{P} = \frac{\frac{\pi D^2}{4} L}{\pi D} = \frac{D}{4}$$

$$\left[i = \tan \theta = \frac{h_f}{L} \right]$$

Pb-① $\frac{f L \cdot V^2}{2gD} = \frac{4 \cdot L \cdot V^2}{D}$

$$C = \sqrt{\frac{8g}{f}}$$

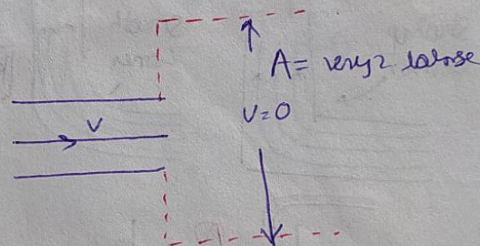
Chezy's coefficient (C) is related to Darcy weisbach friction factor (f)

* ESE 2002

Minor head losses :-

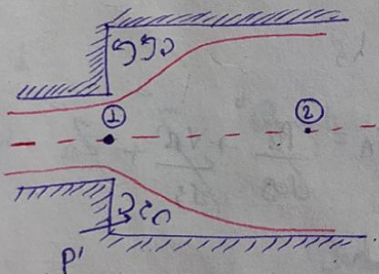
- ① Due to sudden expansion.
- ② At the exit of pipe
- ③ Due to sudden contraction.
- ④ At the entrance of pipe.
- ⑤ Due to pipe bend.

② At the exit point of pipe



$$h_L = \frac{(V-0)^2}{2g} = \frac{V^2}{2g}$$

① Due to sudden expansion :-

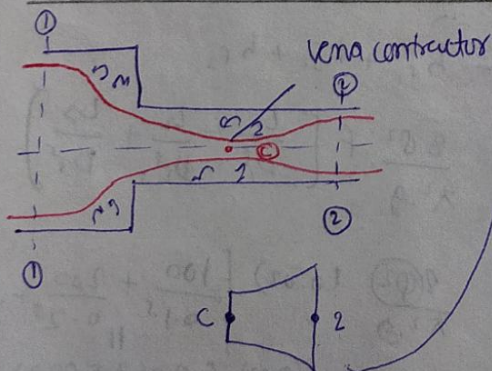


(experiment)

$$P' \approx P_1$$

$$h_L = \frac{(V_1 - V_2)^2}{2g}$$

③ Due to sudden contraction :-



$$h_L = \frac{(V_1 - V_2)^2}{2g}$$

Apply continuity $\Rightarrow$

$$A_1 V_1 = A_2 V_2$$

$$\{ A_1 = C A_2 \}$$

$$(C A_1 / 2) V_1 = A_2 V_2 \Rightarrow V_1 = \frac{V_2}{C}$$

$$h_L = \frac{\left[\frac{V_2}{C_c} - V_2 \right]^2}{2g} = h_L = \left(\frac{1}{C_c} - 1 \right)^2 \frac{V_2^2}{2g}$$

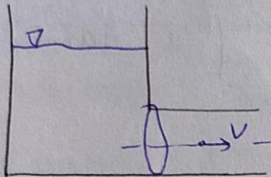
If C_c is not given

$$\left[\frac{1}{C_c} - 1 \right]^2 = 0.5$$

then,

$$h_L = \frac{0.5 V_2^2}{2g} \quad \star \star \star$$

④ At the entrance of pipe:



$$h_L = \frac{0.5 V^2}{2g}$$

⑤ Due to pipe bend:

(Sharp)

$$h_L = \frac{K V^2}{2g}$$

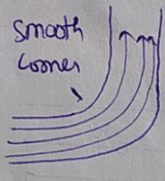
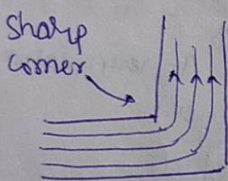
- Effect $\star \star \star$

K = coefficient of pipe bend



Effect $\star \star \star$

Radius of curvature:



$K \downarrow$



$K \uparrow$

Determine % error in discharge if minor head loss neglected.

Apply energy eqn: b/w ① and ②

$$E_B = E_A + h_f$$

$$\frac{P_A}{\rho g} + \frac{V_A^2}{2g} + Z_A = \frac{P_B}{\rho g} + \frac{V_B^2}{2g} + Z_B + h_f$$

$$20 = h_f$$

Neglected minor losses.

$$20 = h_{f1} + h_{f2} + h_{f3}$$

$$20 = \frac{8 Q^2}{\pi^2 g} \left[\frac{L_1}{D_1^5} + \frac{L_2}{D_2^5} + \frac{L_3}{D_3^5} \right]$$

Given,

$$K_{bend} = 1$$

$$f = 0.02$$

$$12090.33 = Q^2 (20625000)$$

$$20 =$$

$$\frac{8 (Q^2)}{\pi^2 g} (0.02) \left[\frac{100}{0.15^5} + \frac{200}{0.25^5} + \frac{100}{0.15^5} \right] = 34118.16 \text{ (Pump)} (20625000)$$

$$Q = 24.22 \times 10^{-3} \text{ m}^3/\text{s}$$

Actual-

$$20 = h_{f_1} + h_{f_2} + 0.5 \frac{V_1^2}{2g} + \frac{V_1^2}{2g} + \frac{V_1^2}{2g} + \frac{(V_1 - V_2)^2}{2g} + 0.5 \frac{V_3^2}{2g} + \frac{V_3^2}{2g}$$

$$20 = \left(\frac{f_1 L_1 V_1^2}{2g D_1} + \frac{f_2 L_2 V_2^2}{2g D_2} + \frac{f_3 L_3 V_3^2}{2g D_3} \right) + \frac{2.5 V_1^2}{2g} + \frac{(V_1 - V_2)^2}{2g} + \frac{1.5 V_3^2}{2g}$$

$$20 = \frac{(0.02) \times 100}{0.1} \frac{V_1^2}{2g} + 20 \frac{V_2^2}{2g} + 21.5 \frac{V_3^2}{2g} + \frac{(V_1 - V_2)^2}{2g}$$

In terms of V_2

$$20 = [22.5(4V_2^2) + 20 + 21.5(4V_2^2) + (4-1)^2] \frac{V_2^2}{2g}$$

$$V_2 = 0.7316 \text{ m/s}$$

Apply continuity

$$A_1 V_1 = A_2 V_2 = A_3 V_3$$

$$\frac{\pi}{4} (0.1)^2 V_1 = \frac{\pi}{4} (0.2)^2 V_2 = \frac{\pi}{4} (0.1)^2 V_3$$

$$(V_1 = 4V_2 = V_3)$$

$$Q_{\text{actual}} = A_2 V_2 = \frac{\pi}{4} (0.2)^2 (0.7316)$$

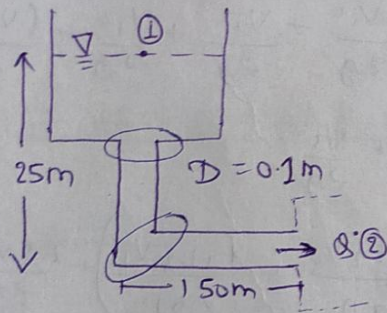
$$= 22.98 \times 10^{-3} \text{ m}^3/\text{s}$$

$$\text{Error \%} = \frac{Q - Q_{\text{actual}}}{Q_{\text{actual}}} \times 100$$

$$= \frac{24.22 - 22.98}{22.98} \times 100$$

$$= 5.39 \checkmark$$

Pb - (14) Note



Apply energy eq b/w ① and ②

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + Z_2 + h_f$$

$$25 = h_f$$

Neglect minor losses

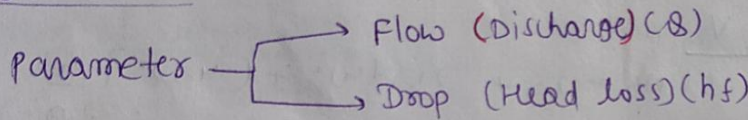
$$25 = \frac{8Q^2 \cdot f \cdot L}{\pi^2 g \cdot D^5}$$

$$25 = \frac{8Q^2 \cdot (0.02) \cdot (25+150)}{\pi^2 g \cdot (0.1)^5}$$

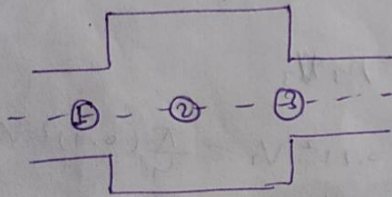
$$Q = 24 \times 10^{-3} \text{ m}^3/\text{s}$$

Different connection -

Flow chart →



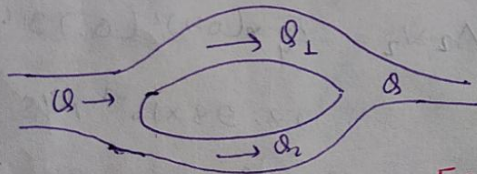
Series →



$Q \rightarrow$ same (d remain same)
(in series)

$$h_f = h_{f1} + h_{f2} + h_{f3}$$

Parallel



$$Q = Q_1 + Q_2$$

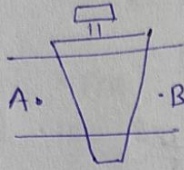
$$h_{f1} = h_{f2} = h_T \quad (h_T = \text{total head loss})$$

[Q will remain same if diameter and velocity whatever]
but in parallel case and total loss will also equal
(Q will be total)

Pb ⑩ [oriented will not used in gas]

Flow Nozzle : having converging section as venturimeter
but not diverging section
42, 5 unit,

Pb ⑪



$$\frac{P_A}{\rho g} + \frac{v_A^2}{2g} + Z_A = \frac{P_B}{\rho g} + \frac{v_B^2}{2g} + Z_B + h_f$$

$$2.5 + 1.5 = \frac{P_B}{\rho g} + Z_B + 0.2$$

$$\frac{P_B}{\rho g} + Z_B = 4.8$$