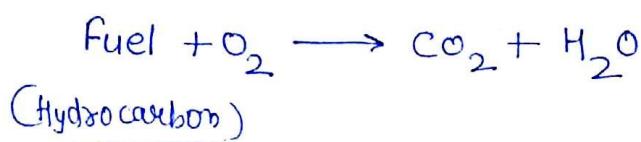
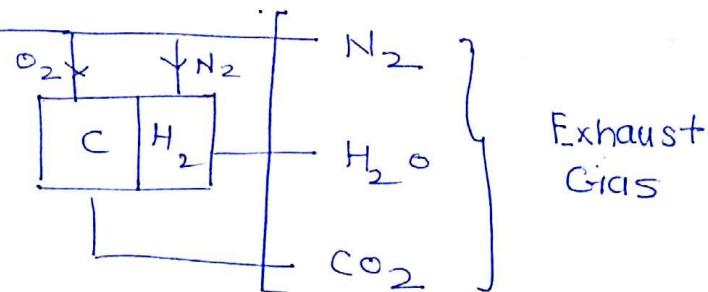


## Combustion in I.C. Engine:-

### Lower calorific value (LCV)

Air	$O_2$	23%	21%
	$N_2$	77%	79%
(By Mass)		(By Volume)	

Air (ideal comb.)



when the steam in the exhaust gas let off freely, than the total heat available due to the combustion of 1 kg fuel is known as the Lower calorific value.

→ The calorific Value of fuel for I.C engine is LCV. (we don't use steam)

Higher Calorific Value :- When the steam in the exhaust gas is ~~not~~ condensed to water and the heat of condensation is utilised for useful heating, the calorific value of fuel is known as HCV.

→ The calorific value of fuel for stationary power plant is HCV. (we use steam)

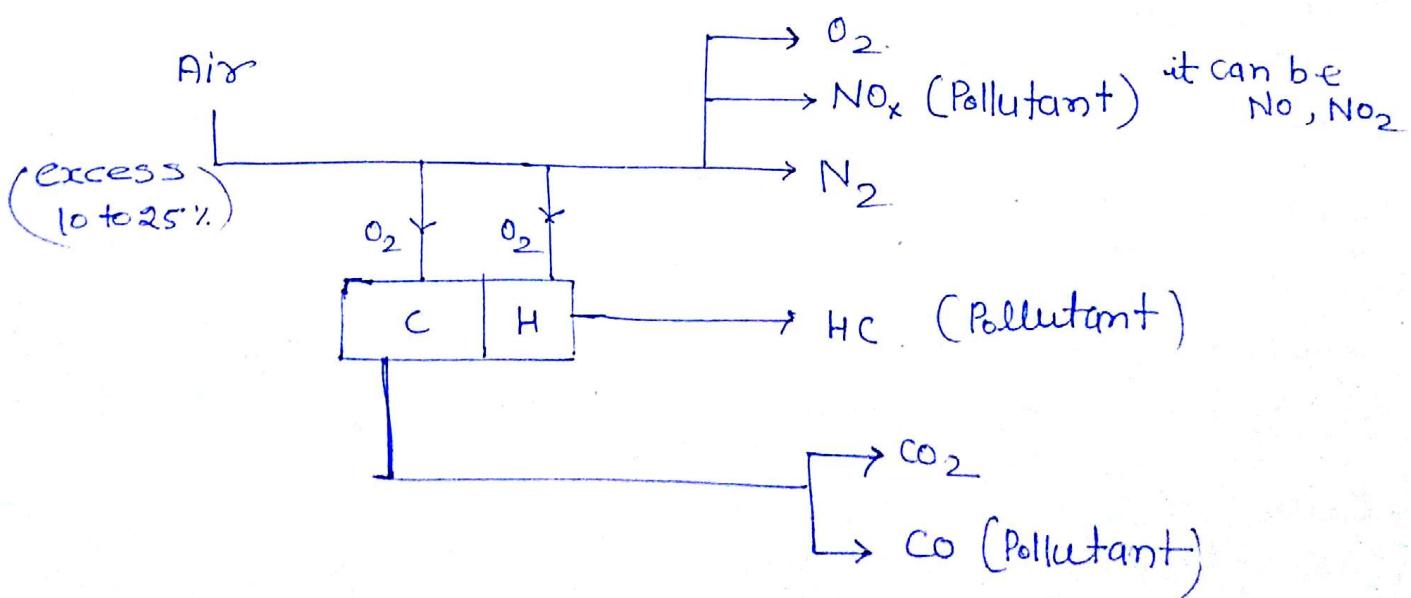
### \* Stoichiometric Air-Fuel Ratio (AFR) :-

when the combustion of Air and Fuel takes place in such manner that there is no free  $O_2$  left ~~knows~~ is any fuel remaining in any form after Combustion is known ideal or perfect combustion. The AFR for such a combustion is known as the stoichiometric AFR.

Stoichiometric AFR ~~by mass~~ → 13.4 to 15

~~by volume~~ → 9 to 9.6

### Actual Combustion :-



### Pollutant :-

- ① CO poisonous, ~~when taken~~ causes giddiness, when taken small quantity, ~~one~~

→ some time can also be fatal.

② HC :- Form photochemical smog when expose to Atmosphere. it causes skin cancer and affects the lungs. Also affects plants and animals.

③ No<sub>x</sub> :- No<sub>x</sub> is form when the temp of the charge is very high

→ other requirement are availability of free O<sub>2</sub> or incomplete combustion of the fuel.

→ when No<sub>x</sub> is expose to atmosphere the effect are the same as HC, besides there formation of ozone(O<sub>3</sub>) at ground level. It is harmful to the extent even underwater creatures are affected.

### EGR :- (Exhaust Gas Recirculation)

EGR is a process that, the exhaust gas is recirculated and mix with fresh AFM that is escaping out of the outlet valve during valve overlap. During this recirculation the temp. of the exhaust gas decrease due to which No<sub>x</sub> is transform into NO & NO<sub>2</sub>.

### Method of Reducing Pollutant:-

- ① slightly excess air(min 10%).  $Q = \frac{1}{m c t} \downarrow \rightarrow \text{No}_x \text{ effec} \downarrow \text{ temp low}$
- ② slightly lesser compression Ratio ( $t \downarrow$ )
- ③ use of thermal converter, it is place in the



path of exhaust Gas. Absorbed air from outside temp. of the thermal converter is higher. It convert HC & CO to  $H_2O$  &  $CO_2$ . Thus the thermal converter is known as a ~~two~~ way converter.

#### ④ Use of catalytic converter.

~~THER~~ (PaPlRho)

(See last page)

Palladium - Co to  $CO_2$

Platinum - HC to  $H_2O$  &  $CO_2$

Rhodium -  $NO_x$  to  $N_2$  &  $O_2$   $\xrightarrow{\text{Reduction}}$  (Reduction)

→ consists of stainless steel box ~~Reduction catalytic of  $NO_x$~~

→ Has alumina inside in form of honey comb.

→ Has palladium, platinum & Rhodium as catalytic agent.

\*  $\Rightarrow$  Orsat apparatus measures  $CO_2$ ,  $CO$ ,  $O_2$  from a ~~fuel~~ unsaturated hydrocarbon gas sample

Equivalent Ratio: - ( $\phi$ )

(Volmetric analysis)

$$\text{Fuel Air Ratio} = \frac{m_f}{m_a}$$

(FAR)

Flue Gas

$CO_2, CO, O_2, N_2$

it is the ratio of FAR under actual condition to FAR under ideal condition (stoichiometric condition)

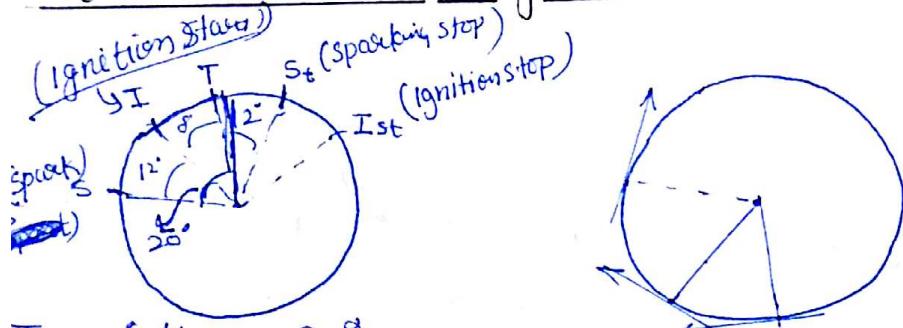
$$\phi = \frac{\text{Actual FAR}}{\text{Stoichiometric FAR}} = \frac{\frac{\text{Actual } m_f}{m_a}}{\frac{\text{ideal } m_f}{m_a}}$$

$$\phi = \frac{\text{Actual } m_f}{\text{ideal } m_f}$$

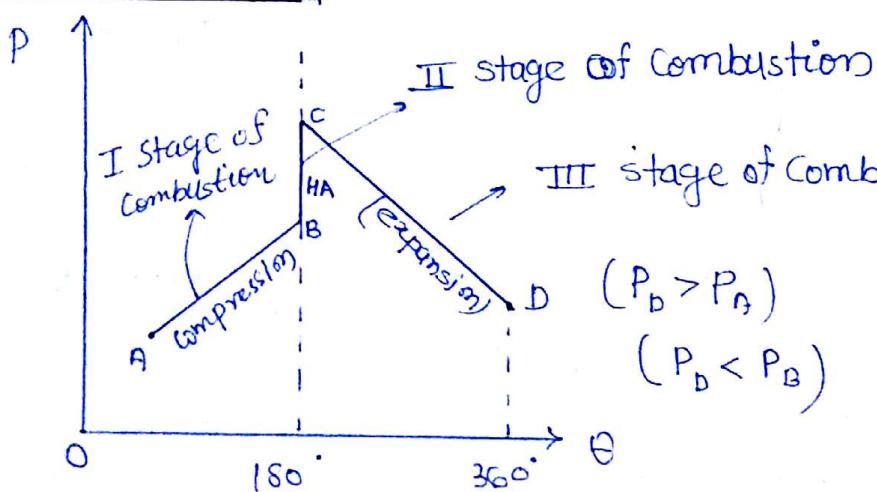
- \* when  $\phi$  is less than 1 ( $\phi < 1$ ), there will be a shortage of the fuel supply. less energy will be liberated from the fuel. The temp. rise will be less the power output will also be less. For Petrol engine, knocking will be less.
- \* when  $\phi > 1$ , the fuel supply will be in excess, temp. of charge will be more high. The power available will also be more. for petrol engeng knocking will also be High.

$\{ \phi \geq 1.4 \text{ less power, No knocking} \}$   
unburn fuel, less air to burn fuel.

### Combustion in SI Engine :-



### Theoretical Diagram

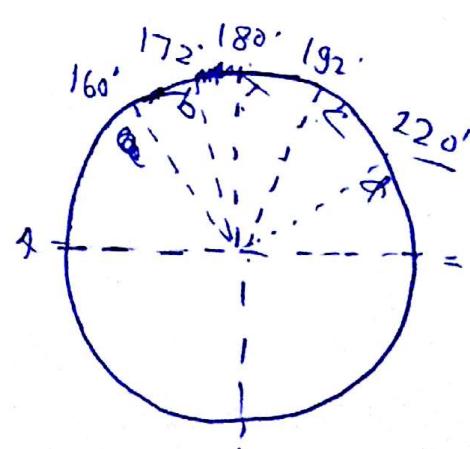


Theoretical diagram  
for 3 stage of combustion

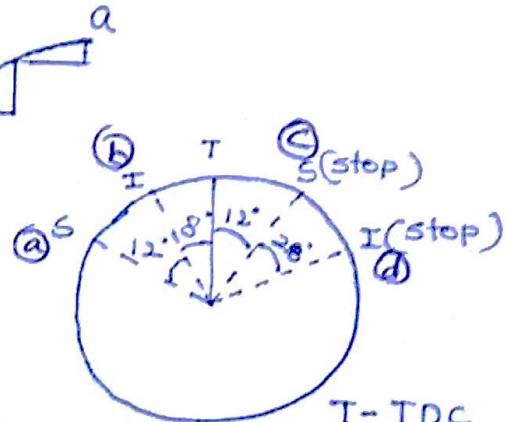
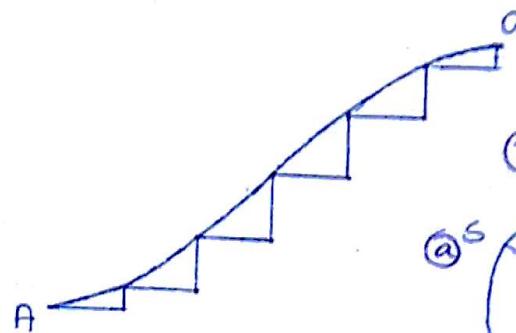
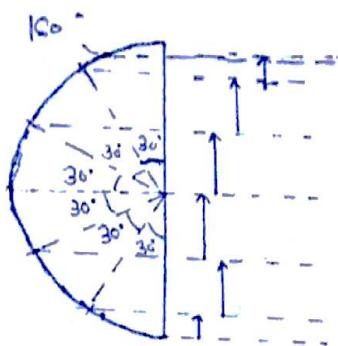
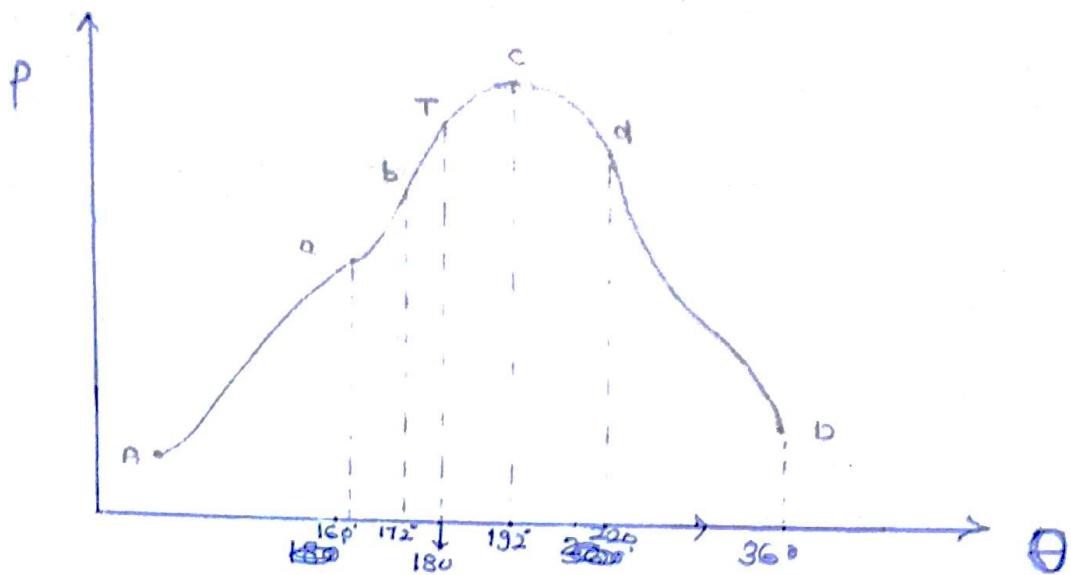
$$\Delta P \propto \theta$$

$$\Delta P = K\theta$$

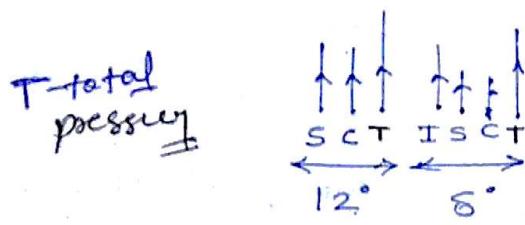
$$y = mx \text{ (St. line)}$$



## Actual Diagram:-



Piston movement as shaft rotates  
(speed varying) { Pressure change according to piston movement }  
after  $160^\circ$



a - S - Sparking start  
 b - I - Ignition start  
 c - S - Sparking stop  
 d - I - Ignition stop

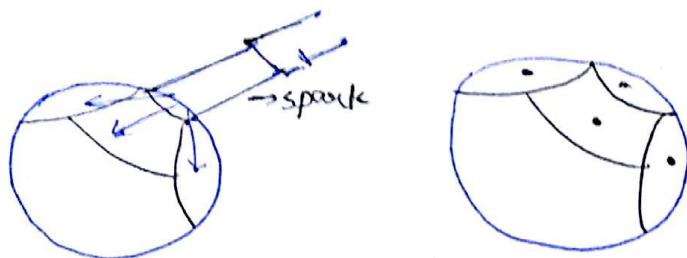
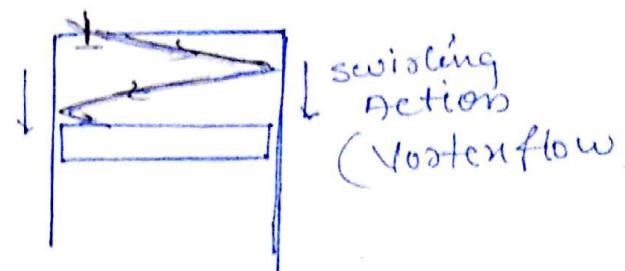
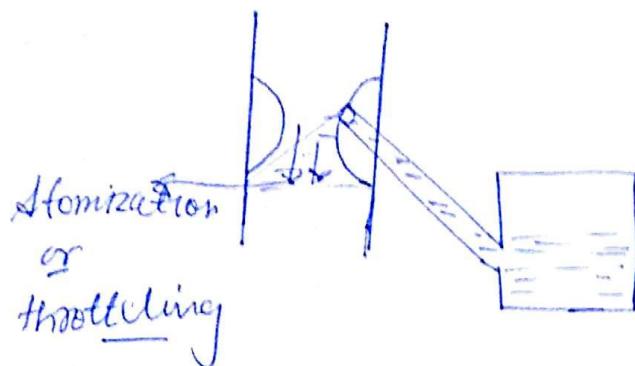
A-a only compression (Normal ↑ in pressure)

a-b 1<sup>st</sup> stage of combustion (only sparking) High ↑ in Pressure

b-T 2<sup>nd</sup> stage of combustion (Very high ↑ in pressure) { Sparking }  
I-T - c - 2<sup>nd</sup> stage of combustion (Gradual ↑ in pressure) { Ignition }

c-d - 3<sup>rd</sup> stage of combustion (Drop in pressure) ↓ { Ignition only }

## Ist stage of combustion:-



+ chemical rxn occurs during heat dissipation temp will not rise  
 Heat dissipated in diff for difference }

when the spark contacts some area of fuel particle, chemical rxn begins in this area. Heat is generated then spread to the other area of the fuel. chemical action begins in the other area also. There is a rise in the temp of the fuel. some heat gets stored in all the different areas.

There is a propagation of Nucleus of the area in which chemical action taking place.

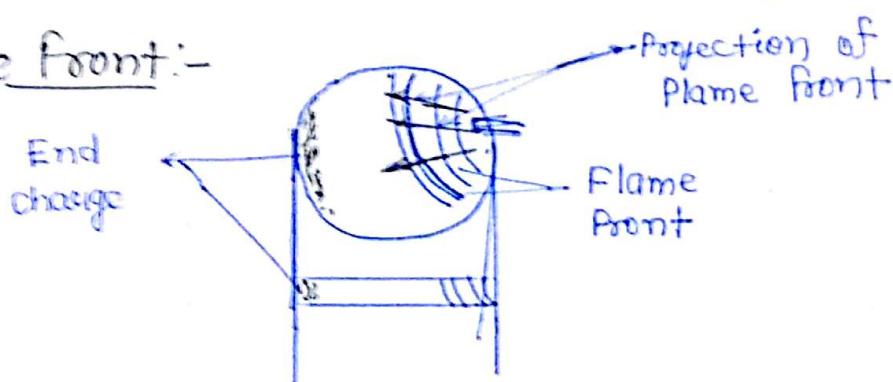
IF the temp of fuel particle is below the self ignition temp. At the end of chemical action of the entire fuel this fuel particle does not burn and will be in circulation along with the gas until exhaust.

At the end of the chemical action if the temp of the fuel particle is equal to or above the self ignition temp. then there is distribution of heat

Collector in the diff area of fuel. There is no change in temp during this heat distribution when equal amount of heat gets collected in the equal diff areas of the fuel ~~the~~ particle, the fuel then get ignited.

- The time taken from the beginning of chemical action to the beginning of the fuel ignition is known as the chemical lag. For petrol engine the chemical lag also known as the ignition lag.
- The first stage of combustion or the chemical lag, is also known as preparatory phase

### Flame Front:-



Flame front:- Due to sparking the nearby fuel particle gets ignited. The imaginary line joining the ignited fuel particle is define as the Flame front

- After deformation of the first flame front, Heat travel from this flame front to the nearby fuel particles and another flame front is formed in the mean time the fuel particle in earlier flame front get completely burn and this flame front vanishes. It will thus appear that the previous flame front has only reach the new position,

This imaginary movement of the flame front is known as the propagation of the flame front.

End charge:- The air fuel mixture at the opposite end of the flame front propagation is known as the end charge.

→ The following thermodynamic event takes place for the end charge during the flame front propagation

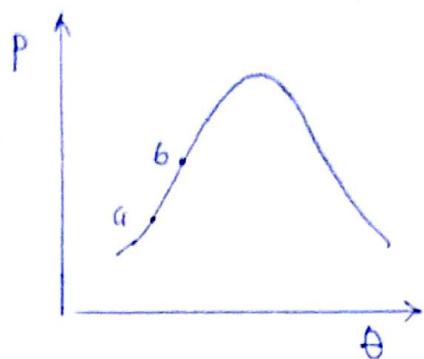
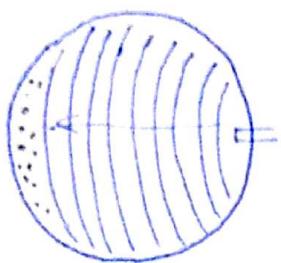
① During flame front propagation the air particle nearby the flame front expands and compresses the end charge, temp. of the end charge increases.

② Due to the continuous increase in the temp. of end charge chemical action begins for the end charge. Temp. of the end charge increases further then -

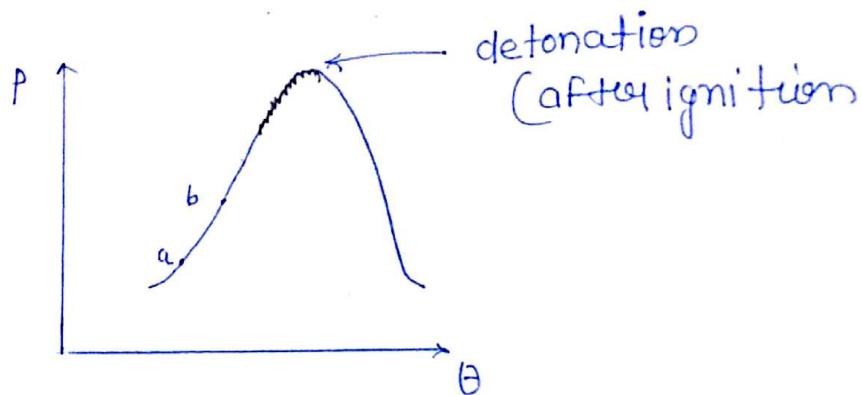
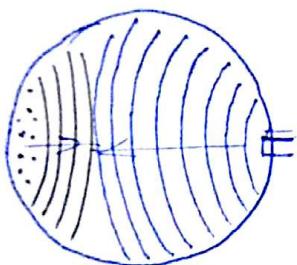
③ There is heat transfer from the flame front to the end charge. Temp of the end charge continuously to increase further.

\* if the temp. of the end charge becomes equal to or above the self ignition temp, a separate flame front form at the end charge. The two flame front move in opposite direction and collide with one another this collision is know as detonation.

→ The following are the disadvantages of Detonation:-



Normal Combustion



Abnormal Combustion

① When detonation take place a wave is developed at the point of collision this wave travel at a very high speed of about 500-5000 cycle/sec. if this wave heat the piston, the piston then starts moving side wise and knock against the cylinder thus detonation and knocking are the same.

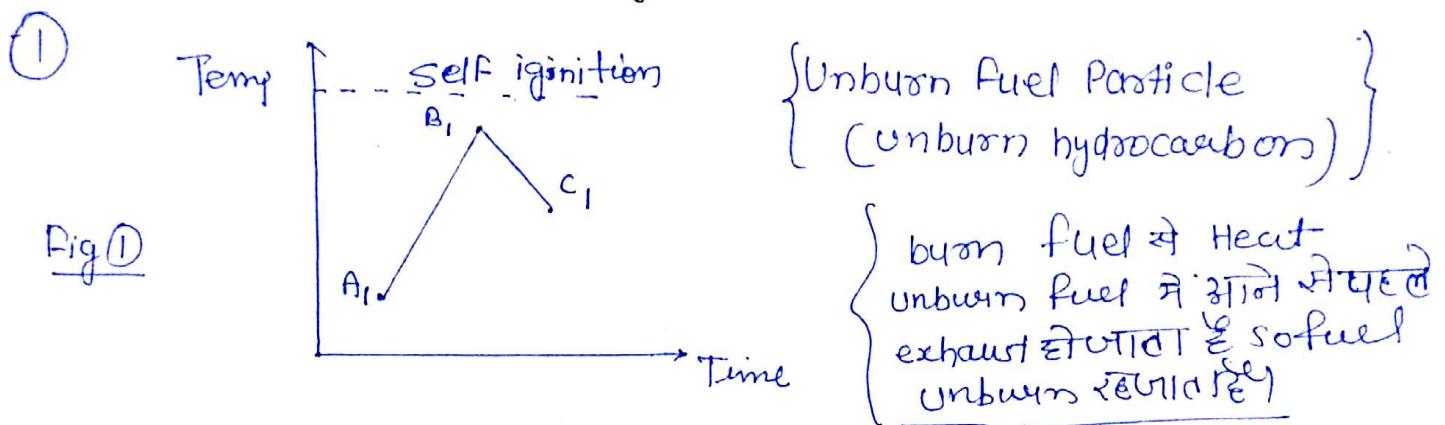
→ Due to knocking a crack may be developed in the piston or in the cylinder

② Due to knocking a loud metallic sound heard known as pinging. Thus detonation and or knocking is also known as pinging.

③ Mechanical vibration are setup inside the engine due to knocking.

④ There is loss of work.

Temp. Time Graph during the chemical action of Fuel.



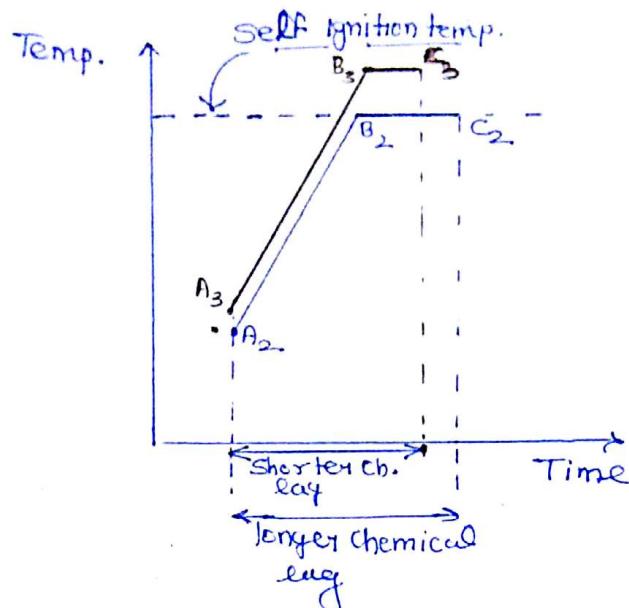
During the chemical action of the fuel particle A<sub>1</sub>, it is found that the rise in temp and the time consumption is given by a straight line. At the end of chemical action the temp. is below the self ignition temp. (point B<sub>1</sub> in the fig.)

→ This fuel particle will remain unburn and is known as the unburn hydrocarbon (UBHC). This UBHC particle will be in circulation with the gases till temp decreases as the temp. will lower down for the gases during expansion after combustion. The lowering of temp. with time is also found to be a straight line (B<sub>1</sub>C<sub>1</sub> in fig.). At one stage this fuel particle will be exhausted out of the engine cylinder (Point C<sub>1</sub>)

(2)

Jump

Fig(2)



- \*  $A_2 B_2$  when temp equal to self ignition
- \*  $A_3 B_2$  when temp more than self ignition
- \*  $B_2 S_2$  Heat distribution time (not intemp.)

Fig(2) shows the temp of the fuel particle ' $A_2$ ' being equal to the self ignition temp at the end of chemical action. For the other fuel particle ' $A_3$ ' the temp is above the self ignition temp at the end of chemical reaction. It may be seen from diagram the chemical lag for fuel particle  $A_2$  is more than that of  $A_3$ .

- It is thus evident, that higher the temp. of the fuel particle (Above the self ignition temp), lesser will be the chemical lag.
- In SI engine, it is obvious that if the temp of the end charge reaches the temp much above the self ignition temp, the chemical lag of fuel particle in end charge will be much less and a separate flame front will be definitely form in the end charge and detonation will take place.

→ It is thus clear that the temp of end charge in SI engine much be less to avoid knocking.

### \* Factors Affecting Detonation in SI engine! -

① Compression ratio:- higher compression ratio will result in the higher temp. of the charge. Tendency to detonate will increase for this region, The compression ratio for SI engine (Petrol engine) is kept as low as between 6 to 10.

→ The effect will be the same when the inlet valve temp. is higher or a supercharger is used.

→ On using a super charger an intercooler must be provide.

② Increasing the Load! - Increasing the load or power will result in more supply of fuel. More energy will be release .The temp. of the charge will be higher thus the petrol engine must be run as less load condition

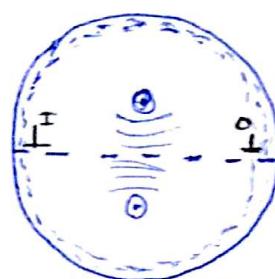
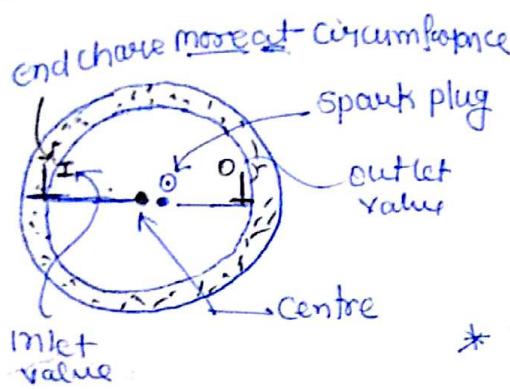
(3) Advancing the Spark! - Advancing the spark will result in more duration of spark. More energy will be receive by the charge .Temp will be higher thus the sparking should not be advance. Sparking may be retarded

~~Knock~~

④ Distance of Flame travel:— The distance covered by the flame front must be such that by the time taken by it should be less than the chemical lag of end charge for this purpose the cylinder diameter of a SI engine is restricted to a maximum 15 cm

(अंग पॉवर बढ़नी ही तो dia. की बढ़ना एवं cylinder use से लेका पड़ा)  
(Knock ही जारी अंग dist ज्यादा हुई हो)

⑤ Location of Spark Plug:-



\* Two spark Plug (Place opposite to center line)

Flame reaches fast at end charge (दूसरी Flame के दूसरों से पहले end charge पहुँच जाएगा)

The spark plug must be located nearest to the centre and away from the inlet valve. The reason is that if inlet valve temp is higher then knocking in SI engine will increase.

⑥ Higher Engine speed!— Increasing the speed will result in fuel balance being setup for the flame front. The flame front will reach the end charge faster in all direction. The tendency to knock will decrease.

Thus the ideal combination to prevent detonation in SI engine is high speed and low load.

(7) Air-Fuel Ratio :- (<sup>see</sup> equivalent ratio  $\phi$ )

(8) Quality of fuel:-

standard fuel

{ Iso octane (zero ~~no~~ knock)  
 { Normal Heptane (~~Readily~~ knocking)

To determine the quality of fuel, It is compared with a standard fuel in the Lab. The standard fuel is a mixture of two other fuel. One is Iso octane (zero knock) and the other is normal Heptane (Readily knocks). If octane in mixture is 70% that matches with the knock quality of petrol, Then petrol is said to have an octane number 70.  $\Rightarrow$  Higher the octane number better is the quality of petrol.

$\Rightarrow$  In India, the octane number of the petrol available is between 70-80. For special quality fuel like extra premium, speed the octane number is between 90-94.

~~Question~~  
IES 2007

What is the value of fuel air ratio in the normal operation range.

- (a) 0.056 to 0.083
- (b) 0.083 to 0.56
- (c) 0.0056 to 0.83
- (d) 0.056 to 0.83

~~Ques~~ air fuel ratio for normal operation speed

$$= \frac{16}{1}$$

0-15 km/h idling speed

$$\text{Fuel air} = \frac{1}{16} = 0.0625$$

15-65 km/h crushing speed

(Normal)

(a)  $\frac{F}{A} = \frac{0.056}{0.083} = 0.675 \times$

(b)  $\frac{F}{A} = \frac{0.083}{0.56} = 0.148 \times$

(c)  $\frac{F}{A} = \frac{0.0056}{0.83} = 0.00675 \times$

(d)  $\frac{F}{A} = \frac{0.056}{0.83} = 0.0675 \checkmark$

choice (D) Match with the fuel air ratio of the normal operating range.

~~v. imp~~  
~~Question~~ In a SI engine Combustion in stage 1 takes 1 millisecond and stage 2 takes 1.5 millisecond when engine runs at 1000 rpm. If Stage 1 time is independent of speed what will be the additional spark advanced when engine speed is double.

- (a) 0°
- (b) 6°
- (c) 12°
- (d) 24°

180°  
6°

Sol<sup>n</sup>

speed (rpm)	Stage 1	Stage 2
1000	1 m sec.	1.5 millisecond
2000	1 m sec.	?

$$\Rightarrow 1000 \text{ rpm} = \frac{1000}{60 \times 1000} \text{ millisecond}$$

$$= \frac{360 \times 1000}{60 \times 1000} \text{ deg millisecond}$$

$$= 6^\circ / \text{msec. } \cancel{\text{Ans}}$$

Correct

~~Ans~~ At 2000 rpm, Angle turned =  $(2 \times 6^\circ) = 12^\circ$

therefore spark advance =  $12 - 6 = 6^\circ$  ~~Ans~~

Question To prevent detonation in SI engine the end charge should have.  
IES 2006

~~a) low temp. b) low density c) long ignition delay~~

$$\rho = \frac{m}{V} \leftarrow \text{const for end charge}$$

↓  
 it mean chemical delay  
 in SI engine

g less  
 so m less  
 So less heat transfer so low temp.

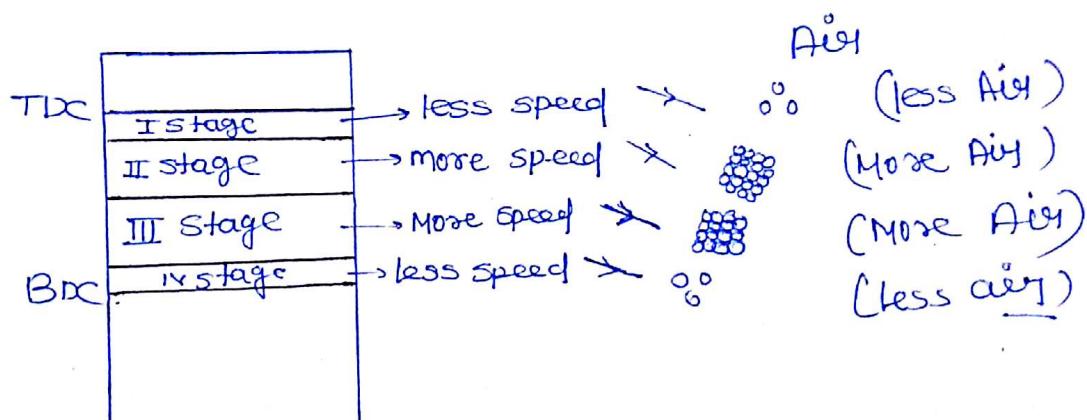


c) low density! - For low density of end charge the mass of the fuel in the end charge will also be less. For less quantity of fuel less heat will be developed during chemical action of fuel particle in the end charge. The temp. of ~~end~~ end charge will also be less.

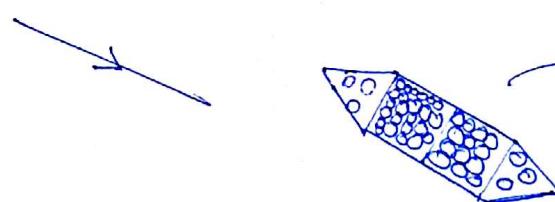
The chemical lag will be more the flame front from the spark plug will reach end charge before deformation of separate flame front in end charge. There for detonation will be prevented.

## Combustion in CI Engine:- (Diesel engine)

### Four stages of suction:-

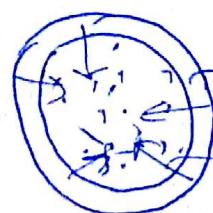
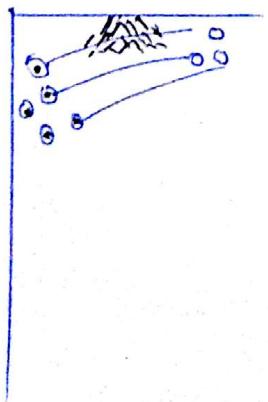


Total Air



Mixing of air will not occur because pressure is maintained

### I Stage of Combustion:-

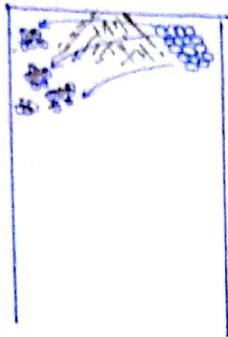


Air (High temp)  
heat Ignite fuel

कम air, fuel के ने पार कर ला दिया।  
एक समय की

- (i) The temp at the start of the I stage of combustion (end of compression) will be close to the self ignition temp
- (ii) less air is circulated during the I stage of combustion hence the fuel supplied during the fuel spray is also very less.
- (iii) when air is crossing the fuel spray, the air and the fuel particle will be physically in contact with one another for some time. This time taken is known as the physical delay
- (iv) Thus, for diesel engine Ignition delay =  $t_{ph}$   
Ignition delay = (Physical delay + Chemical delay)
- (v) Since the temp. of the combustion chamber is close to the self ignition temp, the chemical delay will be longer.
- (vi) The I stage of combustion is known as the delay period
- (vii) The temp. of combustion chamber at the end of the I stage of combustion will be above the self ignition temp.

## II stage of combustion:- (Knocking इन दी जी होती है क्यों)



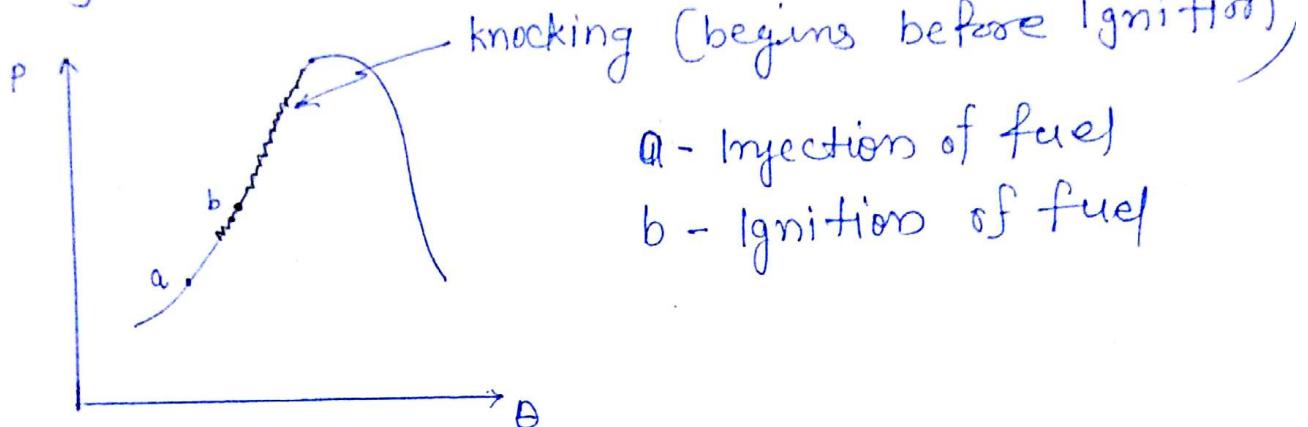
(i) Plenty of air is under circulation. Hence mass of fuel supplied during fuel spray is also large quantity.

- (ii) The temp. of the combustion chamber during the second stage of combustion is above the self ignition temp.
- (iii) Since large mass of air under circulation hence air crosses the fuel spray in diff. batches.
- iv) Due to ~~very high~~ ~~high~~ high temp. of combustion chamber Combustion takes places for all the diff batches of AFM faster. If Combustion is this manner till the last batch of AFM then Combustion is said to be normal or smooth.
- v) On the other hand if a particular batch of air carries less fuel with it, then during chemical action less heat will be generated and the temp of charge will be close to the self ignition temp. The chemical delay will be longer in the main time more batches of AFM will get collected with this batch resulting in the combustion taking place

for all the batches at the same time.

Anomalous amount of energy will be liberated that the piston is unable to handle and knocking occurs.

(vi)



(vii) The second stage of combustion is known as uncontrolled combustion

(viii) The temp at the end of the second stage of combustion is very much above the self ignition temp.

### III Stage Combustion :-

Fuel को पार करती combustion के जायेगा इसलिए



(i) The temp. at the start of 3rd stage of combustion is very much above self ignition temp.

(ii) Plenty of air under circulation during the 3rd stage of combustion. Thus the fuel spreads in large quantity

- (iii) The air crosses the fuel spray in different batches
- (iv) Since the temp. of combustion chamber is much above the self ignition temp hence Combustion takes place for all the batch immediately after crossing the fuel spray.
- (v) Knocking does not takes place during the 3<sup>rd</sup> stage of combustion
- (vi) The III stage of combustion is called **controlled combustion**

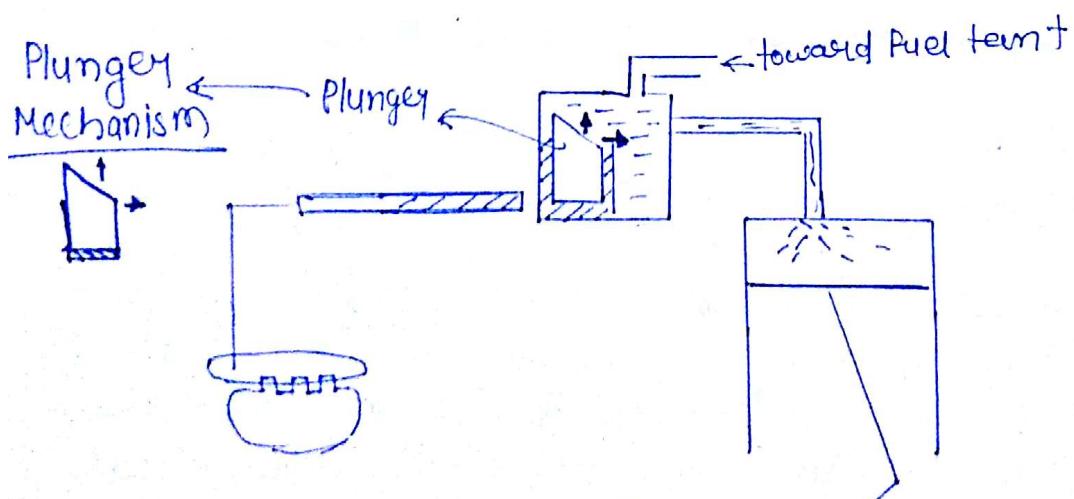
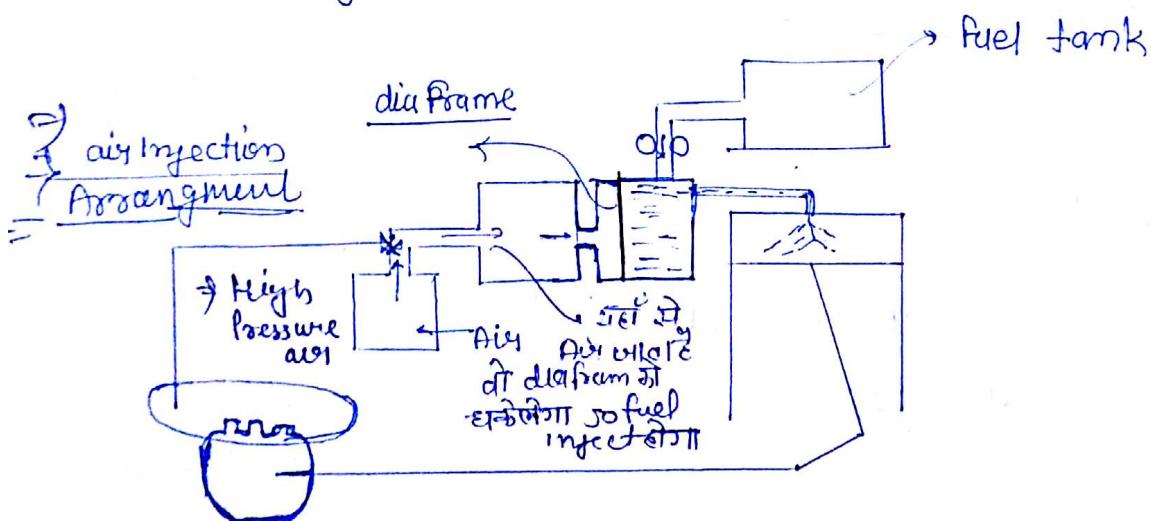
#### IV Stage of Combustion :-

- (i) less air is available for combustion. Hence the fuel supply is also very less.
- (ii) Combustion during the 4<sup>th</sup> stage takes place from the heat transfer after burning of the fuel particle.
- (iii) The 4<sup>th</sup> stage of combustion is known as **after burning**.

## factors Affecting the knocking of CI engine :-

① compression ratio! - Higher compression ratio will result in higher temp. of the charge. Tendency to knock will be less. Due to the reason, the compression ratio of Diesel engine is as high as b/w 16 to 20. → same effect will be the same when inlet valve temp. is higher or Supercharge is used.

## ② Increasing the speed! -



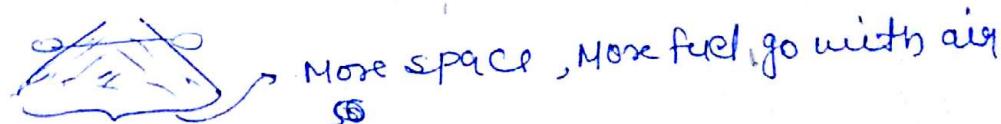
⇒ Solid injection engine (solid acting on fuel)

higher engine speed will result in more fuel supplied during the first stage of combustion. Hence less fuel will be available during the 2<sup>nd</sup> stage of combustion. Tendency to knock will increase. Thus Diesel engine must be run at less speed.

③ Higher Engine load :- Higher engine load will result in more power and more fuel supplied. The temp. of ~~the~~ the charge will be higher. Tendency to knock will decrease.

\* Thus the ideal combination for avoiding knocking in diesel engine is less speed and higher load.

④ Atomisation :- For better atomisation, the Fuel spray will cover more space. The air particles will take more time to cross the fuel spray. The physical lag will be more, Air will carry more fuel with it. Tendency to knock will be reduced.



⑤ Advancing the Fuel Injection :- Advancing the fuel injection will result in lesser compression. Temp. of the charge will be lesser. Tendency to knock will increase. Thus the injection of fuel should not be advanced. It may be retarded.

## ⑥ Quality of Fuel:-

Standard Fuel

- 1) Normal Cetane (zero knock)
- 2)  $\alpha$ -Methyl Napthalene (readily knock)

Higher the cetane number of diesel fuel better will be the anti-knock quality.

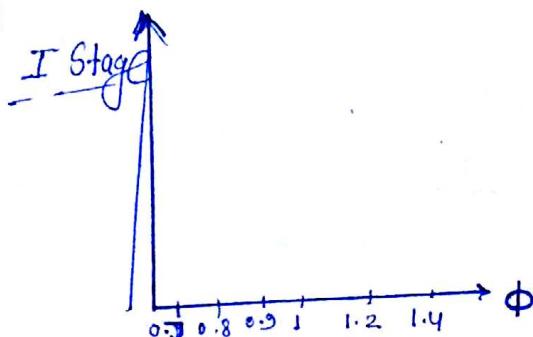
The diesel fuel available in India has a cetane number between 40 to 50.

## Chap-2

Q.1

(i) When the self-ignition temp is increase the temp. of fuel particle must also be increase to the self ignition temp. for combustion the preparatory phase increase.

(ii)



$\left\{ \begin{array}{l} \phi < 1 \\ \text{All fuel burn} \\ \text{Fuel } \uparrow, T \uparrow, \rightarrow \\ \text{कम फैल कम तेंप. (self ignition} \\ \text{के पार)} \\ \text{so chemical delay} \end{array} \right.$

The  $\phi < 1$ , initially, when more fuel supplied at start all the fuel will burn & the temp. of charge will be above self ignition temp the preparatory phase  $\downarrow$ , later when the  $\phi$  is 1.4 or above there will be shortage of  ~~$O_2$~~   $O_2$  thus less fuel will only undergo combustion. temp of charge

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will be close to self ignition. The chemical delay will be longer. More time will be taken for the combustion of the fuel. The preparatory phase will increase.

Q.8

$$\eta = 1 - \left(\frac{V_2}{V_1}\right)^{\gamma-1} \rightarrow \text{constant} \quad (\text{depends only cylinder size})$$

Not on temp.)

$$\eta = 1 - \left(\frac{V_2}{V_1}\right)^{\gamma-1} = 1 - \frac{T_1}{T_2}$$

Ratio will be same

Q.9

Preignition :- Unburn hydrocarbon stick to the inner value of the cylinder, when it is in circulation with gases (After combustion)

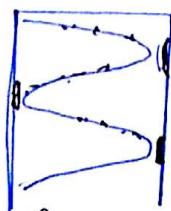


fig.1

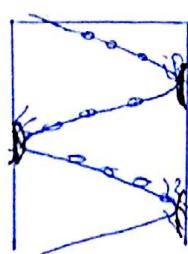


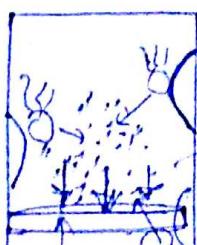
Fig.2

temp घटने पर  
जलने लगा  
पाएंगा।

At the start of the next cycle, fresh AFM will enter in. Some of fuel particle in fresh AFM will contact the UBHC sticking to the cylinder walls. Heat transfer takes place due to which ignition may take place for a few of the fuel particle.

This ignition is known as preignition or misfiring.

Q.10



उत्तर  
को Heat निकालता  
होती Piston motion  
को oppose करता

Ans ③ → The heat released from the ignited fuel will be taken by the air particle and the air inside the cylinder will expand in all directions. The air expanding action would collide with piston moving upward during compression stroke. Due to this there is loss of work, under worst condition the entire work that is