

6. COMPRESSION MEMBERS

⊙ Principle Rafter -

normally used in roof truss as a main strut.

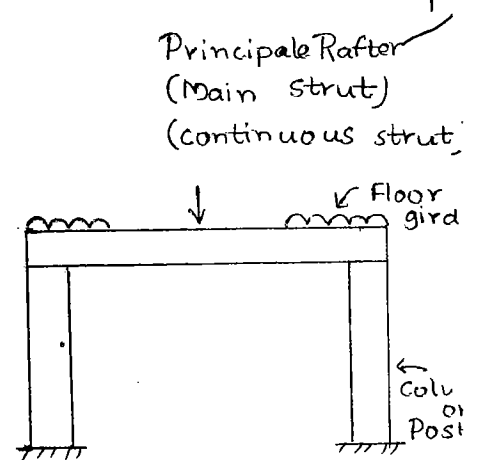
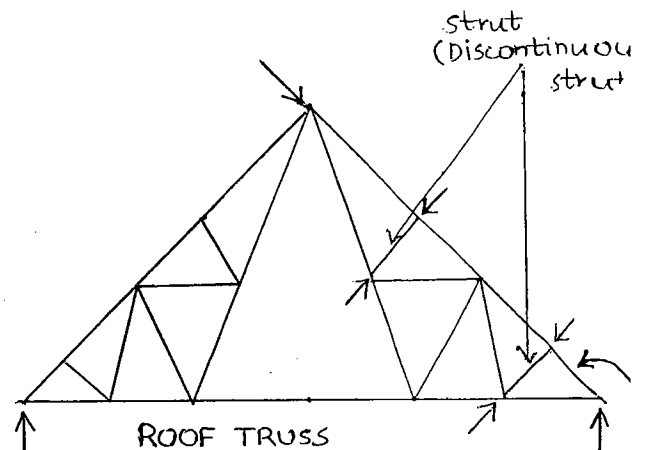
⊙ Strut -

generally used in roof truss (or) in a bracing system.

⊙ Column (or) Post (or) Stanchion -
(steel column)
used in industrial building to support floor (or) floor girders.

⊙ Boom -

Principle compression member in a crane system.

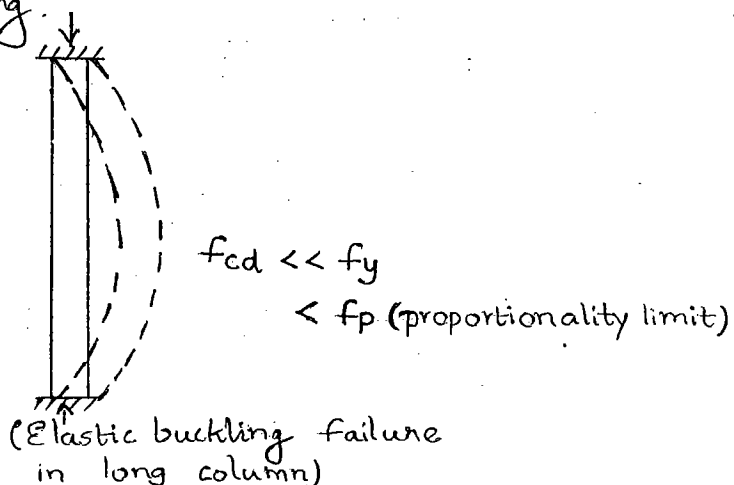
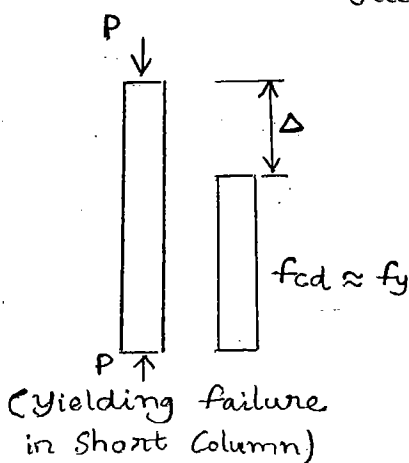


→ Types of Failures in a Compression Member

(i) Short Column or short strut - Normally fail by yielding or crushing of a material.

(ii) Intermediate Column or Intermediate strut - generally fail by inelastic buckling.

(iii) Long Column or Long strut - generally fail by elastic buckling.



— Compressive strength of a column or strut is influenced by: (41)

- (i) Initial imperfection in the member (due to handling, transportation & erection).
- (ii) Residual stress in the c/s (developed due to cooling from high temp while moulding to ambient temp.)
- (iii) Eccentricity of loading (creating additional moments)

→ Design Compression Strength of a Member (P_d)

$$P_d = f_{cd} \cdot A_e$$

f_{cd} → design stress in axial compression

A_e → effective sectional area.

© IS 800:2007 proposes multiple curves a, b, c & d based on PERRY ROBERTSON'S Approach.

λ = non-dimensional effective slenderness ratio.

$$\lambda = \sqrt{\frac{f_y}{f_{cc}}} = \sqrt{\frac{f_y \left(\frac{KL}{r}\right)^2}{\pi^2 E}}$$

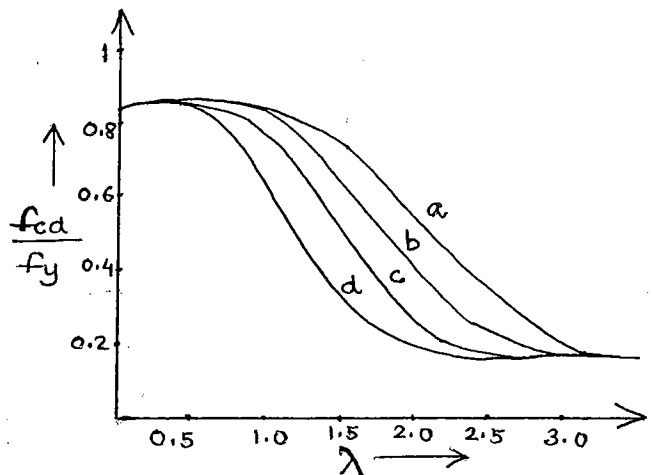
f_{cc} = elastic buckling stress

$$= \frac{\pi^2 E}{\left(\frac{KL}{r}\right)^2}$$

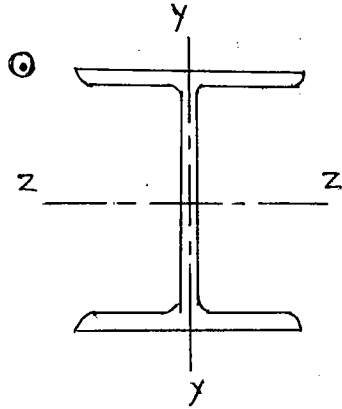
© As per IS 800:1984,

$$P_d = \sigma_{ac} \cdot A_e$$

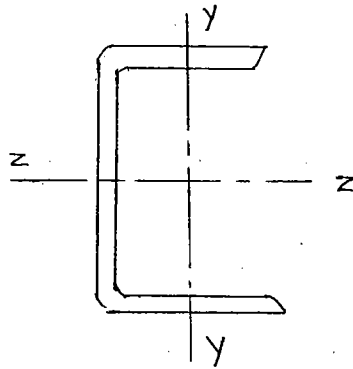
$$\sigma_{ac} = \frac{0.6 f_y f_{cc}}{\left((f_y)^n + (f_{cc})^n\right)^{1/n}} \leq 0.6 f_y \quad (\text{MERCHENT RANKINI FORMULA})$$



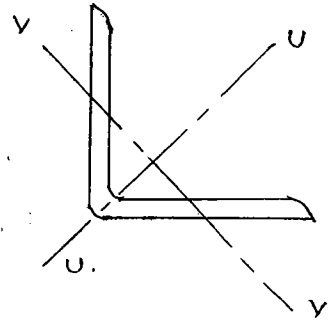
① Class a (tubular sections) members have more compressive strength compared to class c (solid circular sections). Both class a & c members are initially straight and free from eccentric loading. But residual stresses will be more in class c compared to class A as uniform cooling takes place in the interior and exterior of tubular sections.



$$r_{zz} > r_{yy}$$



$$r_{zz} > r_{yy}$$



$$r_{uu} > r_{vv}$$

$$\lambda = \sqrt{\frac{f_y \left(\frac{KL}{r}\right)^2}{\pi^2 E}} \quad \text{for bending}$$

$\therefore \lambda$ will be more, along y-y axis. Higher λ means lower $\frac{f_{cd}}{f_y}$ value from the curve.

$$\therefore (P_d)_{z-z} > (P_d)_{y-y}$$

Economic sections or best sections are those in which

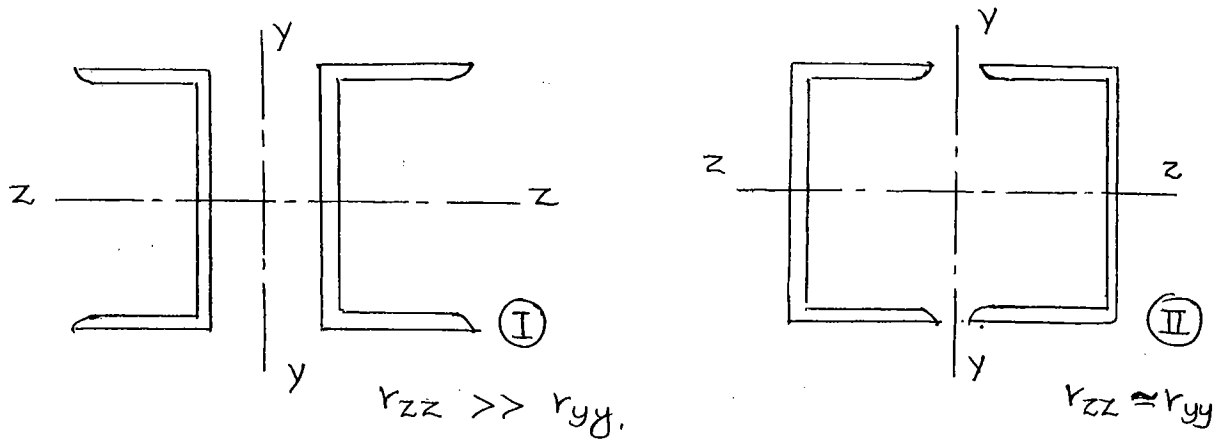
$$(P_d)_{zz} = (P_d)_{yy} \quad (\text{tubular, circular sections})$$

* Perry Robertson's Equation

$$f_{cd} = \frac{f_y / \gamma_{mo}}{\phi + (\phi^2 - \lambda^2)^{0.5}} \leq \frac{f_y}{\gamma_{mo}}$$

$$\lambda \rightarrow \text{non dimensional effective slenderness ratio} = \sqrt{\frac{f_y}{f_{ec}}} = \sqrt{\frac{f_y \left(\frac{KL}{r}\right)^2}{\pi^2 E}}$$

I^{na} section will be subjected to eccentric loading wrt y axis. However for I^{st} section, loading will be CG of section.



For same area of section, II^{nd} section will be more economical and efficient than I^{st} section.

⊙ For struts, tacking bolts, or tacking welds or tacking rivets are used to connect different sections.

⊙ For columns, bracings or lacing systems are used.

* Connecting Systems for Built up Columns are:

- Lacing System (preferred for eccentric loaded column)
- Batten system (generally used for axially loaded columns)

* Lacing System

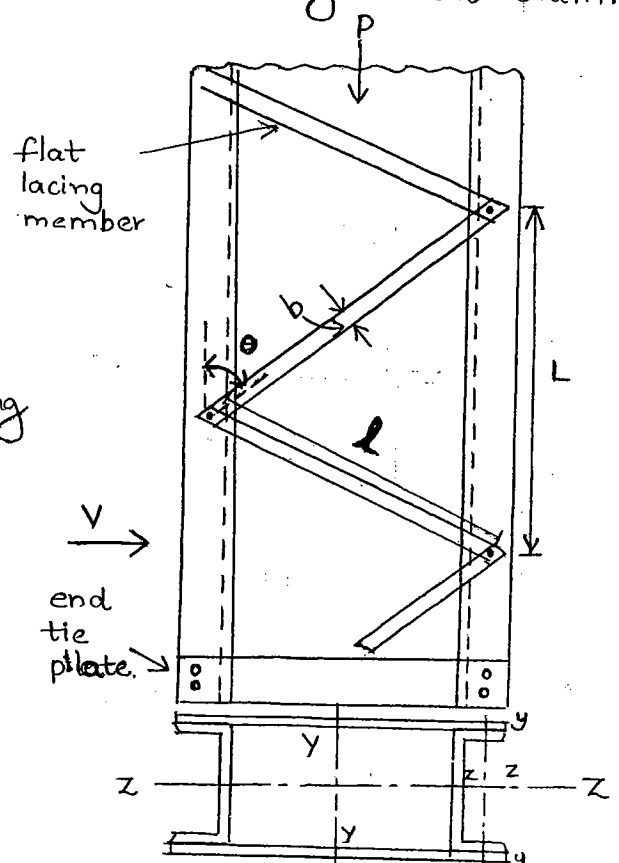
- Flats, angles, channels or tube sections are mostly used as section for lacing member.

$\theta \rightarrow$ angle of inclination of lacing with longitudinal axis.

$L \rightarrow$ spacing of lacing member

$b \rightarrow$ width of flat lacing

$l \rightarrow$ length of lacing member.



Step 3: Select a trial section with approximate area equal to area required with higher minimum radius of gyration.

Step 4: Based on end condition, KL and effective slenderness ratio (KL/r) and f_{cd} of a trial section may be calculated.

$$\text{Step 5: } (P_d)_{\text{trial}} = (f_{cd})_{\text{trial}} \times A_e.$$

If $P \leq (P_d)_{\text{trial}}$; section is safe. Otherwise same procedure is repeated with higher c/s area.

* Limiting Slenderness Ratio.

- ⊙ A compression member subjected to compressive loads resulting from DL + LL combination.
- ⊙ A member subjected to compressive loads resulting from wind load or earthquake loads, WL/EQL
- ⊙ For compression flange of beam.

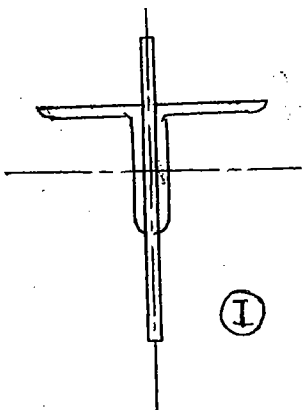
Limiting λ

$$\leq 180$$

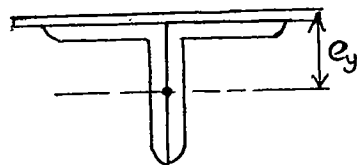
$$\leq 250$$

$$\leq 300$$

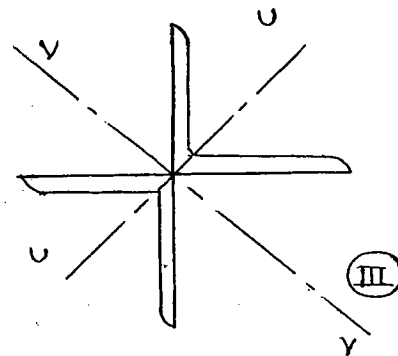
→ Built up Sections (Built up Columns)



Ⓘ



Ⓢ



Out of three sections, IIIrd one is most efficient. as more area thrown away from buckling axis and hence more r .

$\frac{KL}{r} \rightarrow$ effective slenderness ratio.

$KL \rightarrow$ effective length of a ^{compression} member.

$K \rightarrow$ effective length constant ; $L \rightarrow$ unsupported length.

$$\phi = 0.5 (1 + \alpha (\lambda - 0.2) + \lambda^2)$$

$\alpha \rightarrow$ imperfection factor.

Buckling Class	a	b	c	d
α	0.21	0.36	0.49	0.76

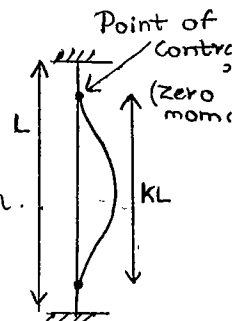
It considers —

- Eccentricity of a load.
- Initial straightness.
- Residual stress present in c/s

$\rightarrow$ Effective Length of a Compression Member: (KL)

KL depends upon :—

- Type of end condition
- No: of members meeting at a joint location.
- No: of rivets or bolts used at a joint.



* Effective Length of a Column (KL)

Type of End Condition	Effective Length (KL)	Effective length constant (K)
Fixed — Fixed.	0.65L	0.65
Fixed — Pinned (or hinged)	0.80L	0.80
Pinned — Pinned (Hinged — Hinged)	1.0L	1.0
Fixed — Free	2.0L	2.0
Fixed — Moment Roller.	1.2L	1.2

Pinned-moment roller



2.0L

2.0

18th Sept,

THURSDAY

→ Design of Axially Loaded Compression Members.

* Design Requirement for Safety of Compression Members:

- Design compressive load (P) $\leq$ Design compression strength of a member.

- $\left(\frac{KL}{r}\right)_0$ of section $\leq$ Limiting Slenderness Ratio.
(to take care of erection loads)

○ Design of compression member is an indirect method of design as the failure stress, f_{cd} , depends on lot of factors.

$$f_{cd} = f\left(\frac{KL}{r}, \alpha, \phi = \sqrt{\frac{f_y}{f_{cc}}}\right)$$

○ For a tension member, section is dependent on A_g .

For a compression member, section is dependent on A_e & r

For a beam, section is dependent on moment of inertia.

Step 1: Assume design stress in axial compression (f_{cd})

$$f_{cd} = 90 \text{ MPa (for angle struts)}$$

$$= 135 \text{ MPa (for I-section columns)}$$

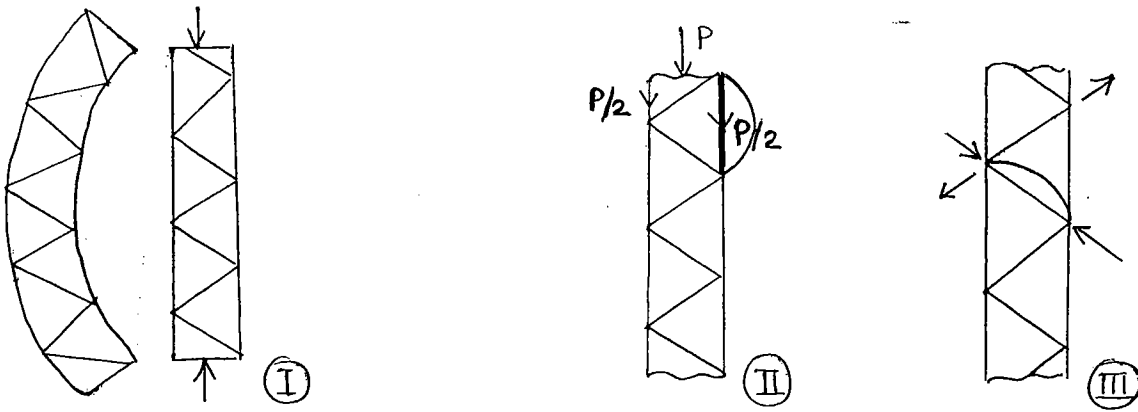
$$= 200 \text{ MPa (for heavily loaded columns)}$$

Step 2: Effective sectional area required for factored compressive load, P is :

$$(A_e)_{\text{req}} = P / (f_{cd})_{\text{assumed}}$$

- Failure of lacing system

45 (4)



① → buckling of whole component

② → local buckling of column component.

③ → local buckling of lacing member

- General Specifications:

⊙ The radius of gyration normal to the plane of lacing not less than parallel to the plane of lacing

$$\text{ie, } r_{yy} \neq r_{zz}$$

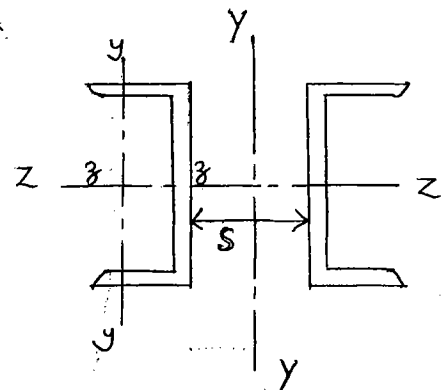
To have economy in design, condition must be

$$r_{yy} = r_{zz} \Rightarrow I_{yy} = I_{zz}$$

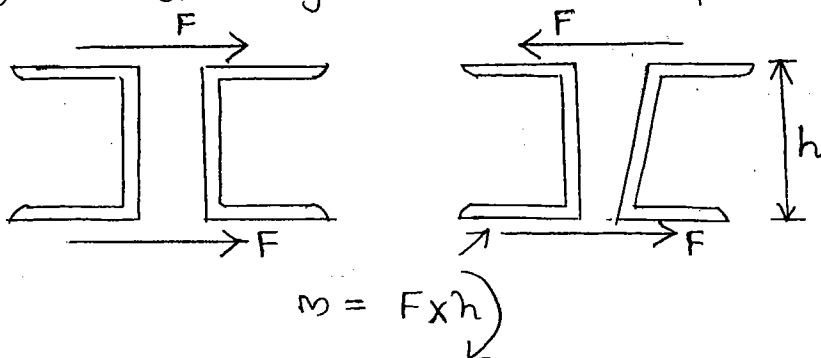
$$I_{zz} = I_{yy}$$

$$2I_{zz} = 2(I_{yy} + Az^2)$$

$$= 2(I_{yy} + A(\frac{s}{2} + c_{yy})^2)$$

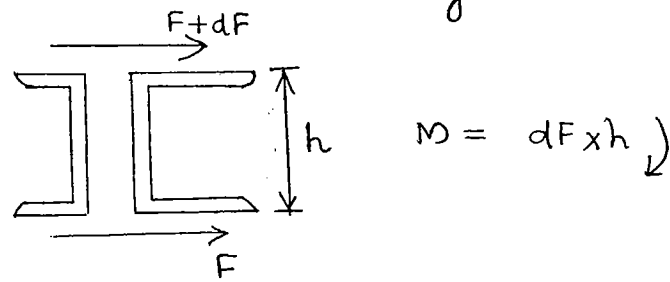


⊙ For single lacing system, lacing on one plane should reflect mirror image to the other plane



$$M = F \times h$$

- There should not be any variation in lacing system



- Design Specifications:

(important)

$$40^\circ \leq \theta \leq 70^\circ$$

Optimum angle for lacing member, $\theta = 45^\circ$ to 50°

If $\theta < 40^\circ$, lacing member may chance to carry some of the the column load and same load may transfer from one column component to another column component like truss member.

If $\theta > 70^\circ$, the tying force carrying capacity of lacing member is decreased.

- Effective slenderness ratio of lacing member should be less than or equal to 145.

$$\left(\frac{KL}{r} \right)_{\text{lacing}} \leq 145$$

Above condition is required to avoid local buckling of lacing member b/w individual column components.

(i) For single lacing system with one bolt or one rivet.

$$\frac{KL}{r} = 1.0l$$

(ii) For single lacing with welds.

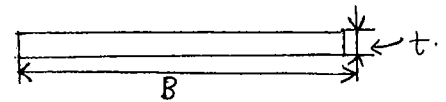
$$0.7l$$

(iii) For double lacing system

$$0.7l$$

where $KL \rightarrow$ effective length of lacing member (Kl)

For flat lacing ($B \times t$)



$$I_{\min} = \frac{Bt^3}{12}$$

$$r_{\min} = \sqrt{\frac{I_{\min}}{A}} = \sqrt{\frac{Bt^3}{12 \cdot Bt}} = \frac{t}{\sqrt{12}}$$

$$\frac{\sqrt{12} KL}{t} \leq 145$$

Min width of flat lacing member, $B_{\min} \approx 3 \times$ shank diameter of bolt.

Shank diameter of bolt	M ₁₆	M ₁₈	M ₂₀	M ₂₂
B (min) (mm)	50	55	60	65

Min thickness of lacing member

$t_{\min} \neq \frac{l}{40}$; for single lacing system

$\neq \frac{l}{60}$; for double lacing member.

$$\frac{L}{r_{\min}^c} \leq 50$$

$\leq 0.7 \times$ effective slenderness ratio of whole built up column $\left(\frac{KL}{r}\right)_0$

whichever is less.

$r_{\min}^c \rightarrow$ min. radius of individual column component.

The above condition is required to eliminate local buckling of individual column component b/w lacings.

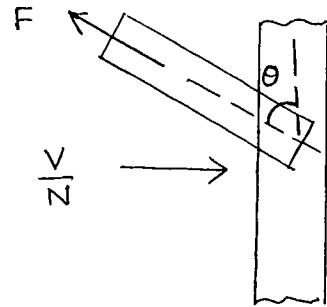
Lacing members ~~may~~ ^{should} be designed for a transverse shear of 2.5% factored column load.

Transverse shear, $V = 2.5 \times \text{factored column load}$

$$V = \frac{2.5 P_u}{100}$$

$$\frac{V}{N} - F \sin \theta = 0$$

$$F = \frac{V}{N} \sin \theta$$



• Design axial force in lacing member (F)

$$F = \frac{V}{N \sin \theta}$$

$$F = \frac{V}{2 \sin \theta} \quad (\text{for single lacing, } N=2)$$

$$= \frac{V}{4 \sin \theta} \quad (\text{for double lacing system } N=4)$$

• Effective slenderness ratio of lacing column should be increased by 5% in order to take care shear deformations due to unbalanced horizontal forces in lacing members.

Effective slenderness ratio, $\frac{KL}{r} = 200$

$$\frac{l}{d} = ?$$

 $KL = 1.0 L$ (Ends are hinged)

$$\frac{KL}{r} = 200$$

$$\therefore \frac{L}{r} = 200$$

$$r = \sqrt{\frac{I}{A}} = \sqrt{\frac{\pi d^4/64}{\pi/4 d^2}} = \frac{d}{4}$$

$$\frac{L}{d/4} = 200 \Rightarrow \frac{l}{d} = \underline{\underline{50}}$$

4. Transverse shear, $V = \frac{2.5P}{100}$

 $P \rightarrow$ Factored load. ; $P_s \rightarrow$ service load.

$$P = \gamma_L \times P_s$$

$$= 1.5 \times 1000 = 1500 \text{ kN.}$$

$$\therefore V = \frac{2.5 \times 1500}{100} = \underline{\underline{37.5 \text{ kN}}}$$

5. Axial force in lacing, $F = \frac{V}{N \sin \theta}$

$$\theta = 45^\circ ; N = 2.$$

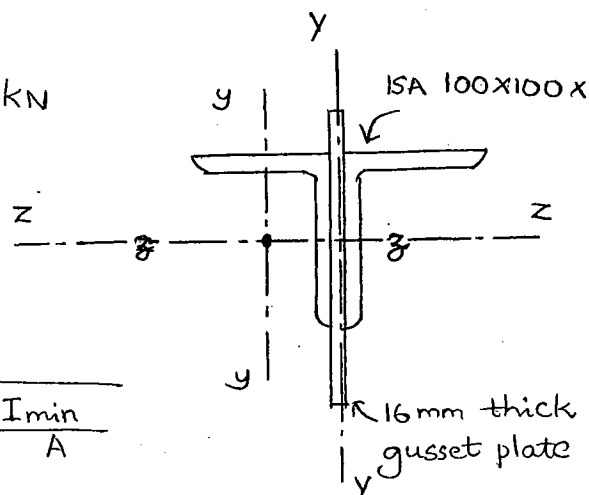
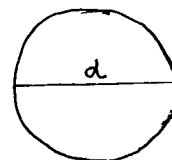
$$F = \frac{37.5}{2 \sin 45} = \underline{\underline{26.52 \text{ kN}}}$$

6. $A = 1903 \text{ mm}^2$

$$I_{zz} = I_{yy} = 177 \times 10^4 \text{ mm}^4$$

$$\text{Min radius of gyration, } r_{\min} = \sqrt{\frac{I_{\min}}{A}}$$

$$I_{\min} = \min \text{ of } I_{yy} \text{ or } I_{zz}.$$



$$I_{zz} = 2(I_{Gz} + Ay^2)$$

$$= 2(I_{zz} + A\bar{y}^2) = 2I_{zz}$$

Gusset plates are provided only at points of junctions.
 $\therefore$ I & A of gusset plates are not included

$$I_{yy} = 2(I_{yy} + A(\bar{x}/2 + C_{zz})^2)$$

$$\Rightarrow I_{zz} < I_{yy} \quad \therefore I_{\min} = I_{zz}$$

$$r_{\min} = \sqrt{\frac{2I_{zz}}{2A}} = \underline{\underline{30.6 \text{ mm}}}$$

8. $I_{\min} = \min I_{zz} \text{ or } I_{yy}$

$$I_{zz} = 2(I_{zz} + A\bar{y}^2)$$

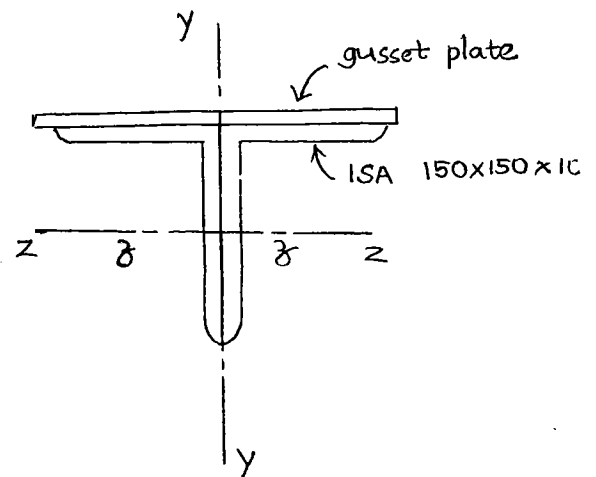
$$= 2I_{zz}$$

$$I_{yy} = 2(I_{yy} + A\bar{x}^2)$$

$$= 2I_{yy} + 2A(40.8)^2$$

$$I_{\min} = I_{zz}$$

$$r_{\min} = \sqrt{\frac{I_{\min}}{A}} = \sqrt{\frac{2I_{zz}}{2A}} = \sqrt{\frac{6.335 \times 10^6}{2921}} = \underline{\underline{46.57 \text{ mm}}}$$



9. $r_{\min} = \sqrt{\frac{I_{\min}}{A}}$

$$I_{\min} = \min \text{ of } I_{vv} \text{ or } I_{uu}$$

$$I_{uu} = 2(I_{uu} + A\bar{v}^2)$$

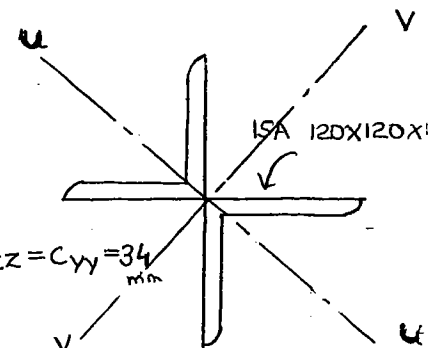
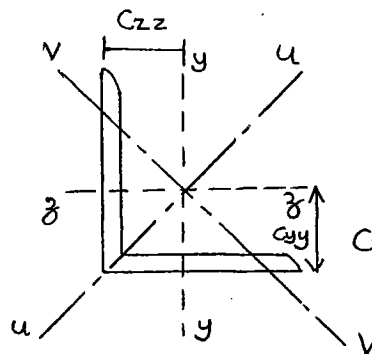
$$= 2(584 \times 10^4 + 2750 \times 0^2)$$

$$= 1168 \times 10^4 \text{ mm}^4$$

$$I_{vv} = 2(I_{vv} + A\bar{u}^2)$$

$$= 2(151 \times 10^4 + 2750 \times 2312)$$

$$= 1573.6 \times 10^4$$



$$A = 2750 \text{ mm}^2$$

$$I_{zz} = I_{yy} = 368 \times 10^4$$

$$I_{uu} = 584 \times 10^4$$

$$I_{vv} = 151 \times 10^4$$

$$\bar{u}^2 = C_{zz}^2 + C_{yy}^2 = 34^2 + 34^2 = 2312$$

$$I_{min} = I_{uu} = 1168 \times 10^4 \text{ mm}^4.$$

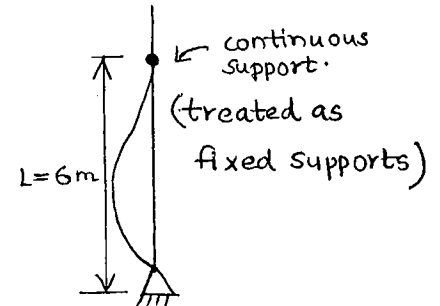
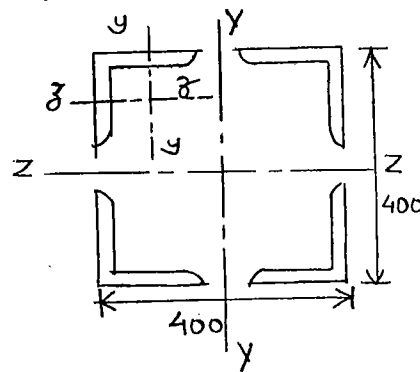
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$$r_{min} = \sqrt{\frac{I_{uu}}{A}} = \sqrt{\frac{1168 \times 10^4}{2 \times 2750}} = \underline{46.08 \text{ mm}}$$

10. $A = 1903$, $I_{zz} = I_{yy} = 177 \times 10^4 \text{ mm}^4$.

$$C_{yy} = C_{zz} = 28.4 \text{ mm}$$

$$r_{yy} = r_{zz} = 30.5 \text{ mm}.$$



P-47 : 6th Point.

Buckling about any axis of built-up member $\rightarrow$ class C

Design axial compressive load, $P = f_{cd} \times A_e$.

f_{cd} = Design axial compressive stress

A_e = Effective sectional area = $4 \times 1903 \text{ mm}^2$

$$f_{cd} = \frac{f_y / \gamma_{mo}}{\phi + (\phi^2 - \lambda^2)^{0.5}} \leq \frac{f_y}{\gamma_{mo}}$$

Assume grade of steel, Fe 410

$$\Rightarrow f_y = 250 \text{ MPa}, \gamma_{mo} = 1.10$$

Built up members - class C $\Rightarrow$ Imperfection factor, $\alpha = 0.49$

λ = non dimensional effective slenderness ratio

$$= \sqrt{\frac{f_y \left(\frac{KL}{r} \right)^2}{\pi^2 E}}$$

$KL = 0.8L$ (one end hinged & other end fixed).

$$= 0.8 \times 6000 = 4800 \text{ mm}.$$

MI of built up section: $I_{zz} = I_{yy} = (I_{Gg} + Ay^2) 4$

$$= 4 \left(177 \times 10^4 + 1903 \times \left(\frac{400}{2} - 28.4 \right)^2 \right)$$

$$= 231.227 \times 10^6 \text{ mm}^4$$

$$r_{\min} = r_{yy} = r_{zz} = \sqrt{\frac{I_{zz} \text{ or } I_{yy}}{4A}}$$

$$= \sqrt{\frac{231.227 \times 10^6}{4 \times 1903}} = 174.28 \text{ mm.}$$

Effective slenderness ratio = $1.05 \frac{KL}{r}$ (5% ↑ for built up column).

$$\lambda = \sqrt{\frac{f_y \left(\frac{KL}{r} \right)^2}{\pi^2 E}} = \sqrt{\frac{250 \times \left(\frac{1.05 \times 4800}{174} \right)^2}{\pi^2 \times 2 \times 10^5}} = \underline{\underline{0.325}}$$

$$\phi = 0.5 (1 + \alpha(\lambda - 0.2) + \lambda^2)$$

$$= 0.5 (1 + 0.49(0.325 - 0.2) + 0.325^2)$$

$$= 0.5834$$

$$f_{cd} = \frac{f_y / \gamma_{m0}}{\phi + (\phi^2 - \lambda^2)^{0.5}} = \frac{250 / 1.10}{0.58 + (0.58^2 - 0.325^2)}$$

$$f_{cd} = 214.33 < \frac{250}{1.1} \text{ N/mm}^2$$

$$P_d = f_{cd} \times A = 214 (4 \times 1903)$$

$$= 1628 \times 10^3 \text{ N} = \underline{\underline{1628 \text{ kN}}}$$